



# The dropwise condensation heat transfer characteristics of CNT/OTS layered surface associated with non-condensable gas effect

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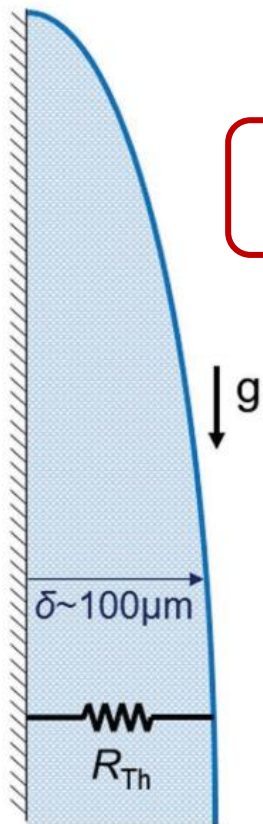
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2020. 12. 16 ~ 18

# Introduction

## Dropwise condensation

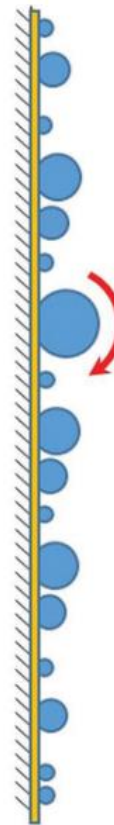
**Filmwise**



Hard to contact  
Steam on surface

Large thermal  
resistance

**Dropwise**



Easy to contact  
Steam on surface

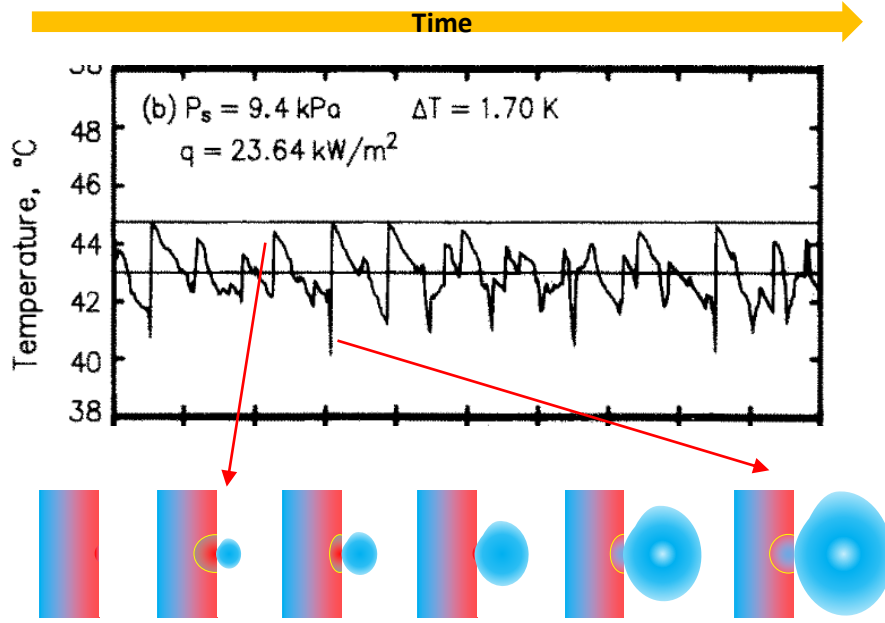
Droplet sweeping  
and re-nucleation

# Introduction

## Dropwise condensation (Constriction resistance)

- **Constriction resistance** is caused by the bending of the heat flux lines in substrate
- It causes non-uniform heat flux and surface temperature on dropwise condensation

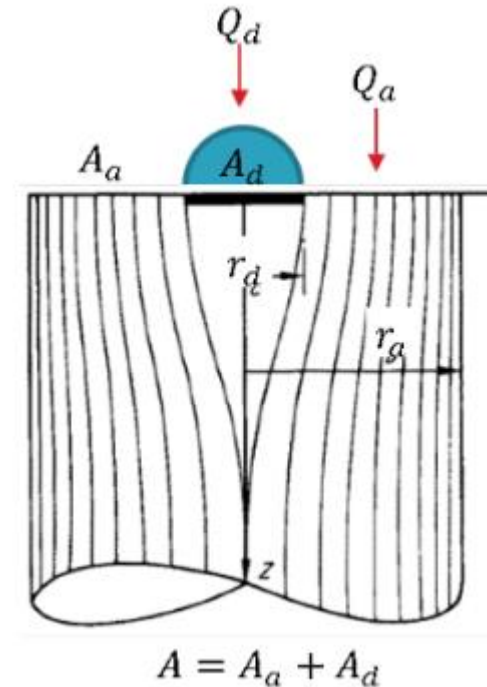
Temperature fluctuation



Droplet distribution



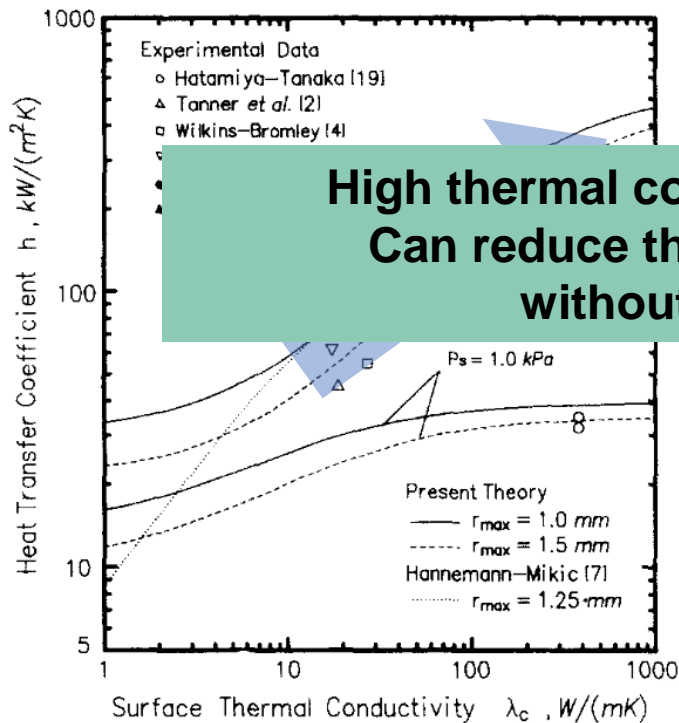
Heat flux line



# Introduction

## Dropwise condensation (Constriction resistance)

- **High thermal conductive material** can reduce the constriction resistance
- The substrate material is often limited by its mechanical strength and integrity
- **Changing the substrate material is impractical** in many industrial applications



**High thermal conductive material coatings  
Can reduce the constriction resistance  
without substrate change**

surface

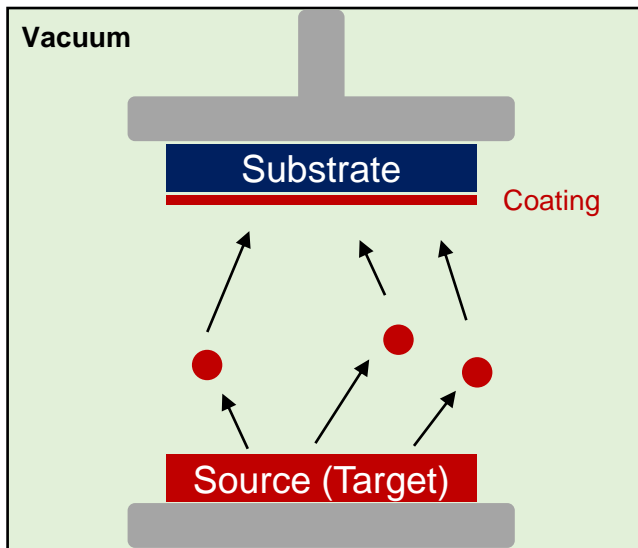
**High heat transfer coefficient**

# Introduction

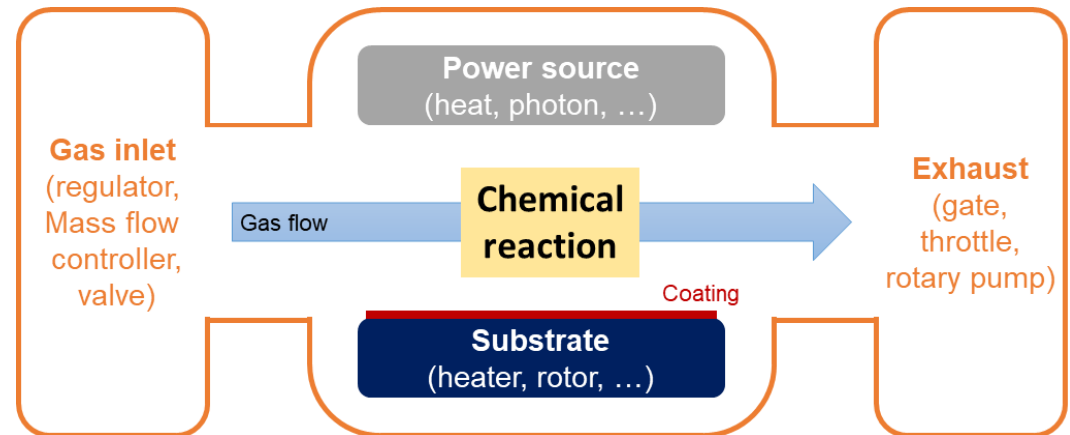
## Surface modification technique

- Surface modification can change the surface material without altering substrate
- **CVD and PVD are well known surface modification techniques**
- However, it needs high temperature or vacuum condition
- It is difficult to apply the commercial heat transfer tubes such as heat exchanger

**PVD**



**CVD**



## The objective of this study

- Useful surface modification technique(**Layer-by-layer**) was selected.
- High thermal conductive material(Multi-walled carbon nanotube) was deposited
- **Condensation heat transfer experiment** was conducted on modified surface

High thermal conductive coatings



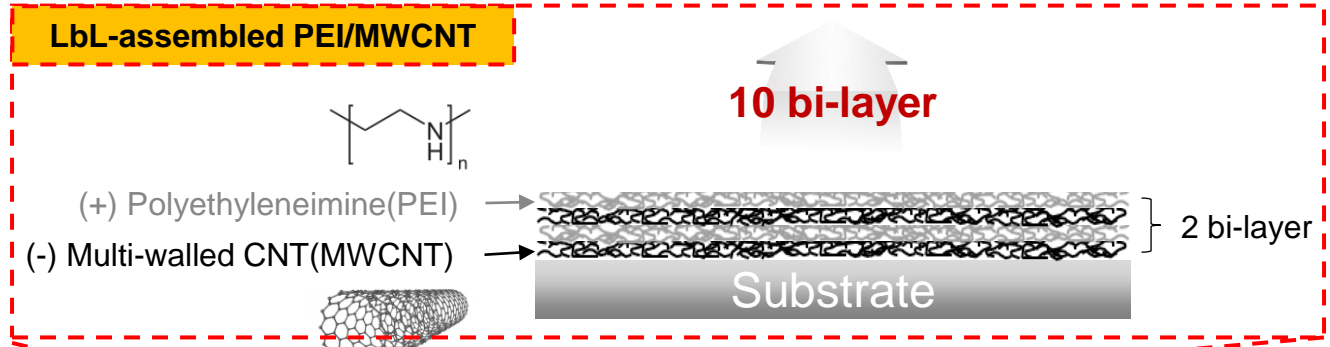
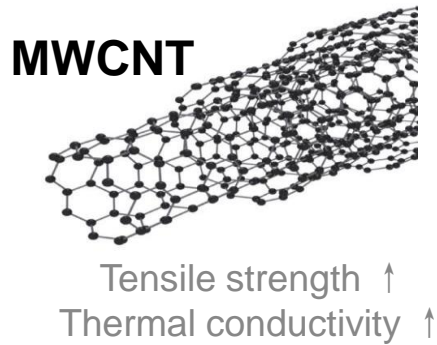
Condensation heat transfer experiment



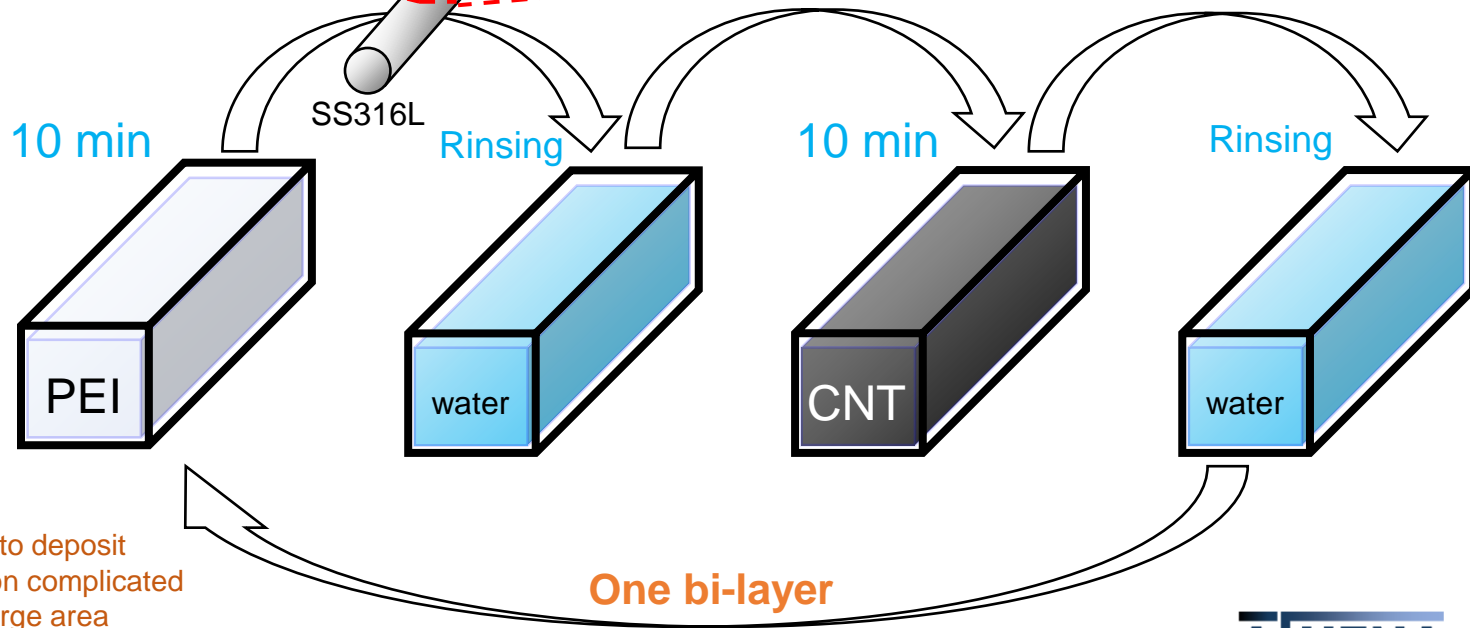
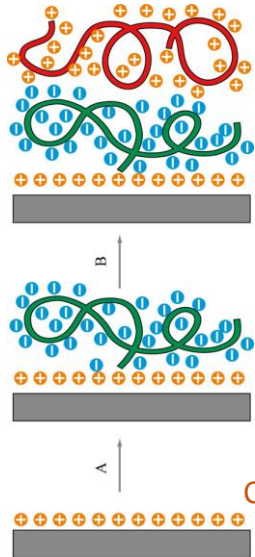
Discussion of the constriction resistance

# Surface modification

## LbL assembled MWCNT coating



### Layer-by-Layer(LbL)

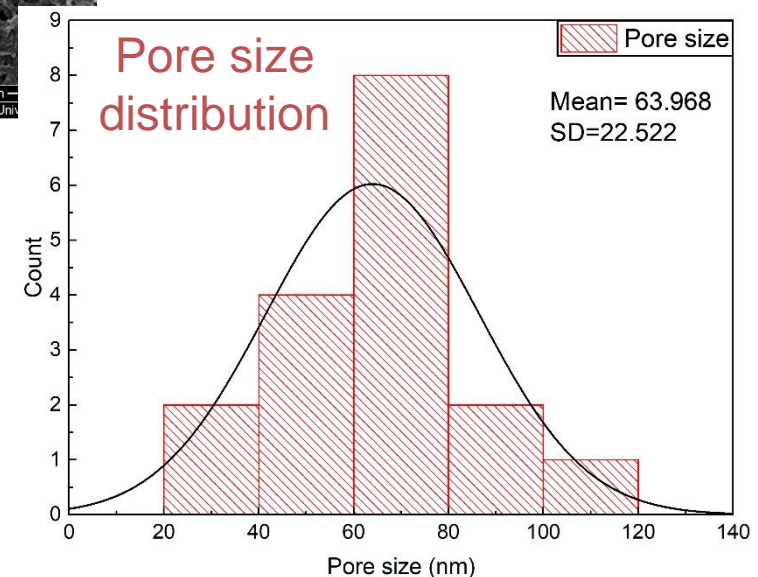
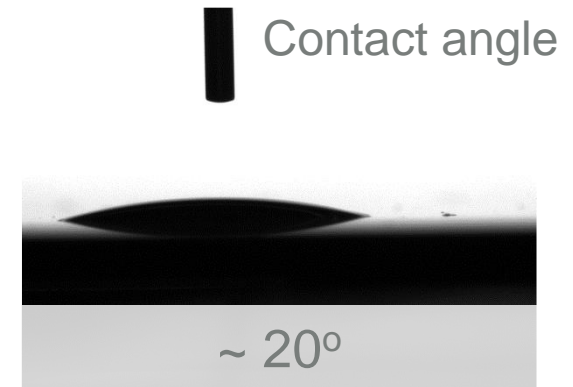
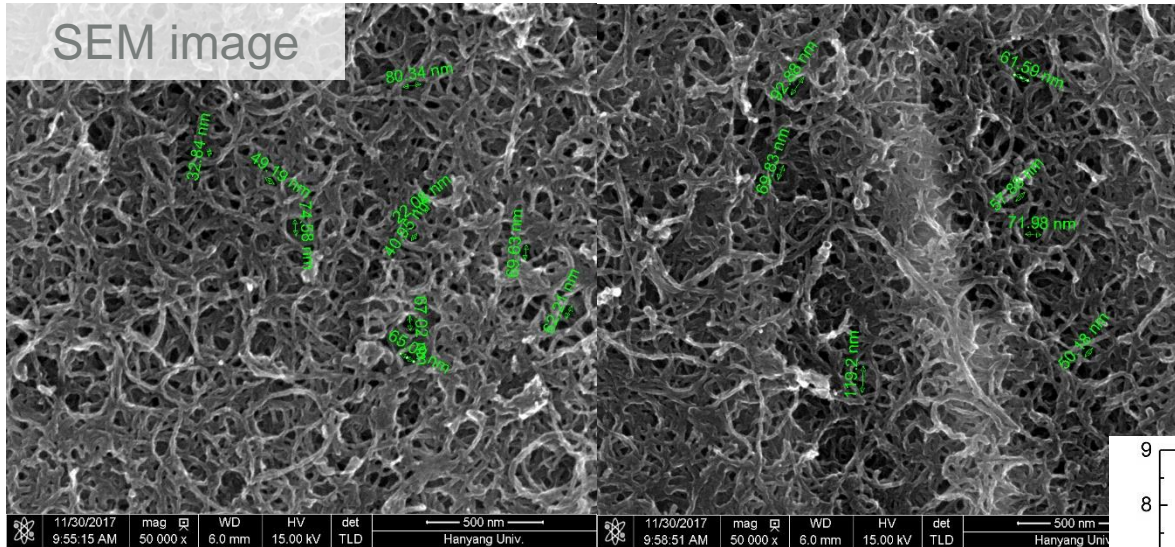


Easy to deposit  
Capable on complicated  
or large area

One bi-layer

# Surface modification

## LbL assembled MWCNT coating



Coating material	CNT
Bi-layer	10
Pore size	$\sim 64$ nm
Thickness	$\sim 760$ nm (ref)
Contact angle	$\sim 20^\circ$

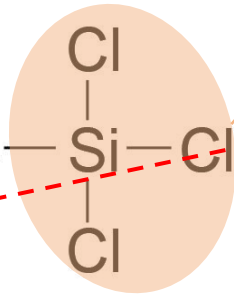
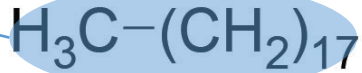


# Surface modification

## Hydrophobic coating

### Octadecyltrichlorosilane(OTS)

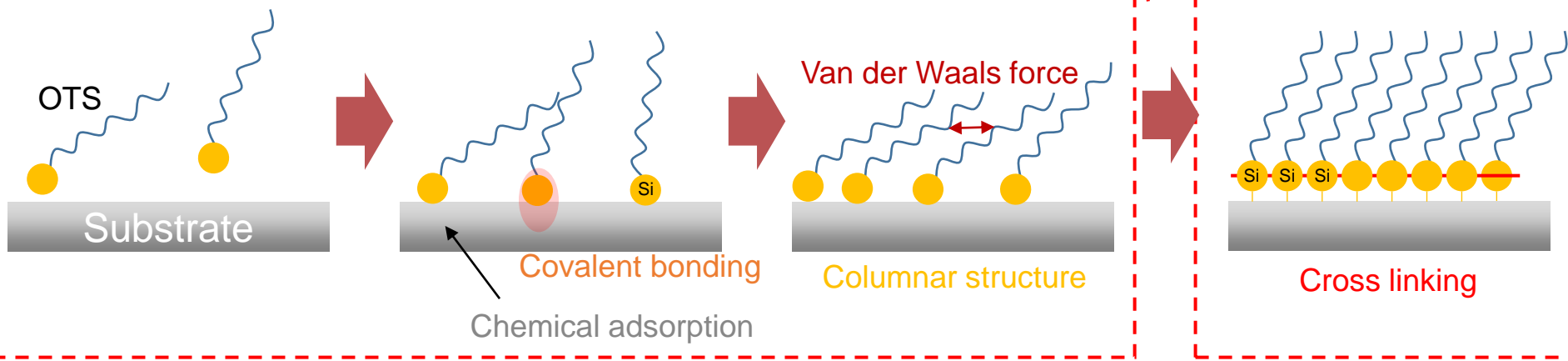
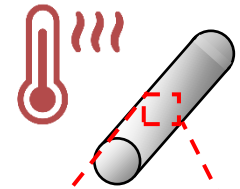
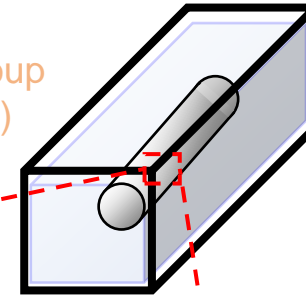
Methyl group  
(Hydrophobic)



Head group  
(Silane)

~ 2 h dipping

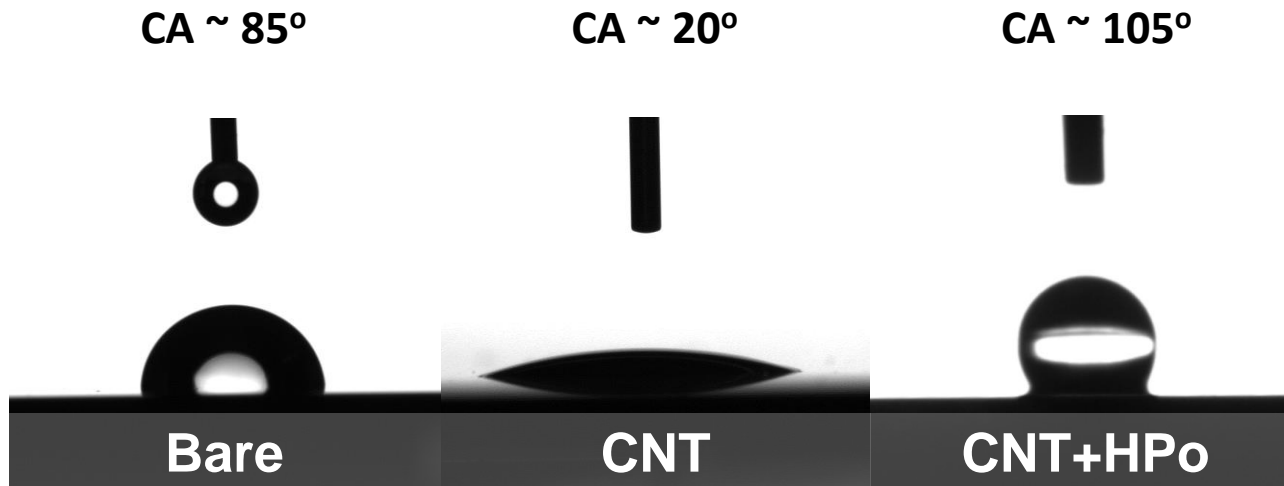
Annealing



# Surface modification

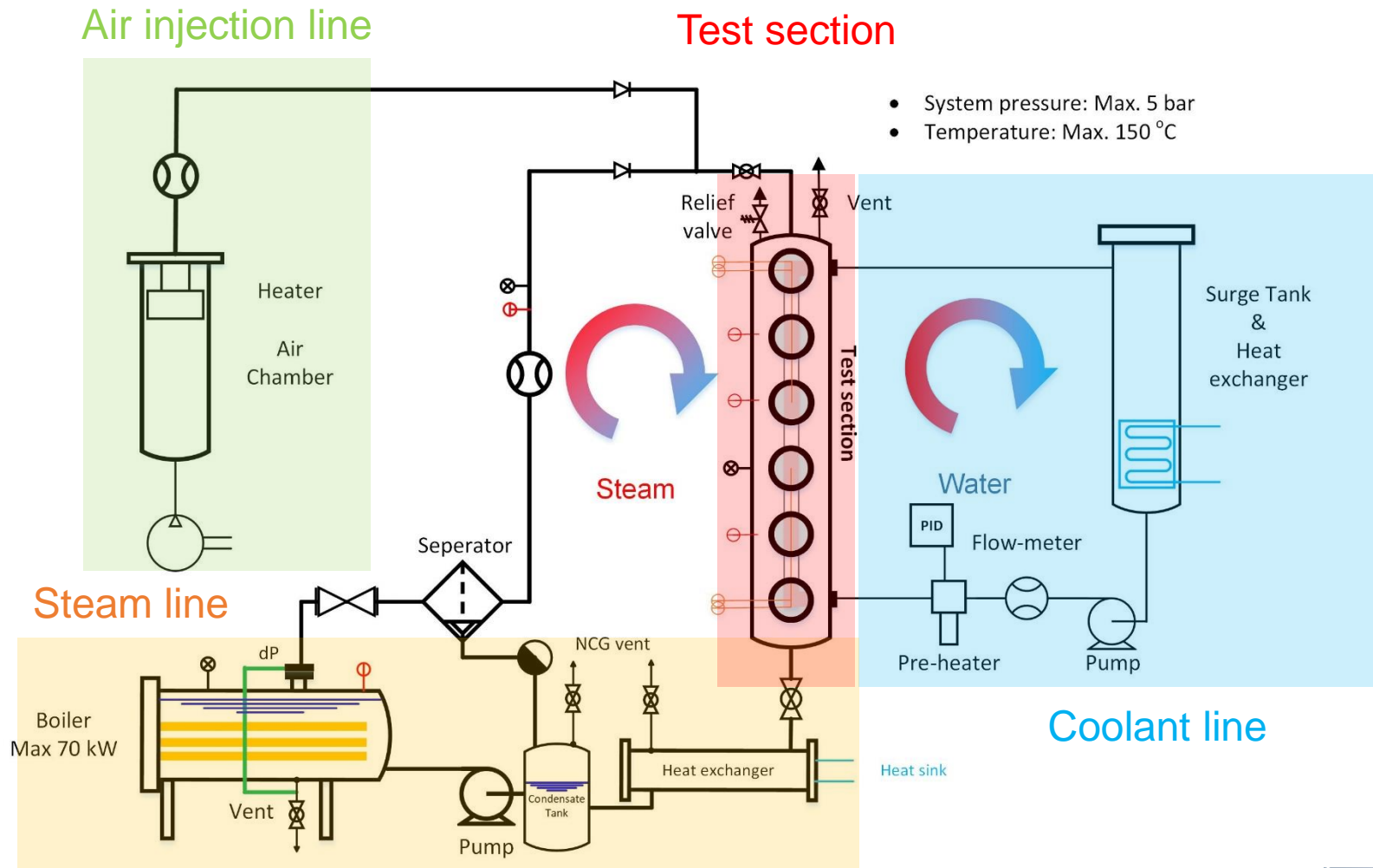
## Surface preparation

- Three surface were prepared for the condensation heat transfer experiment
- The substrate was selected to commercial stainless-steel (Bare)
- LbL-assembled MWCNT surface(CNT) cannot promote the dropwise condensation
- Hydrophobic coatings on the CNT surface was deposited (CNT+HPo)

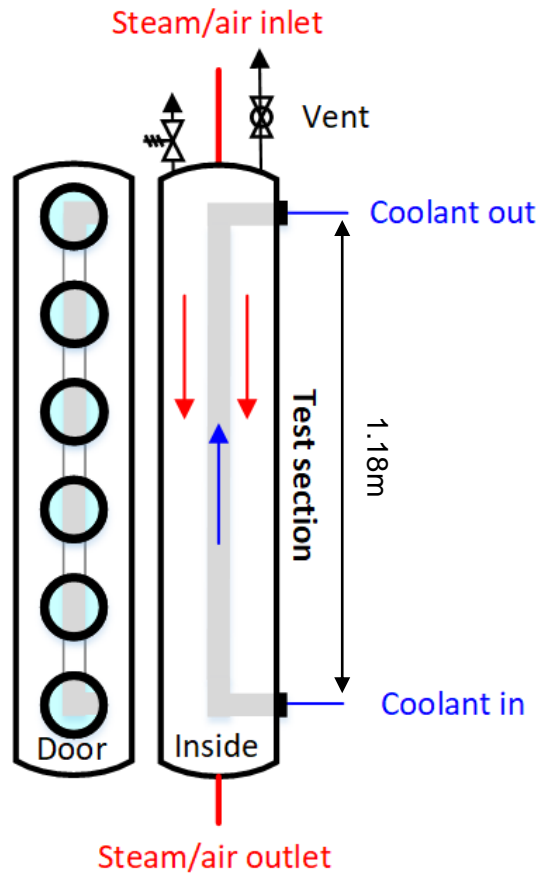


# Method

## Experimental set up (Test Loop)



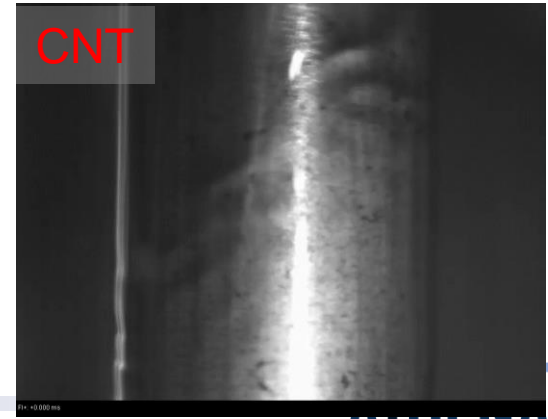
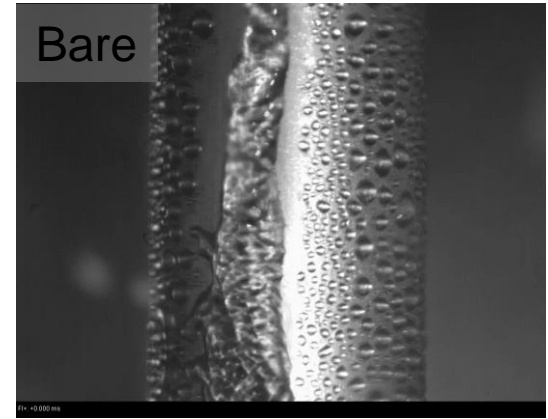
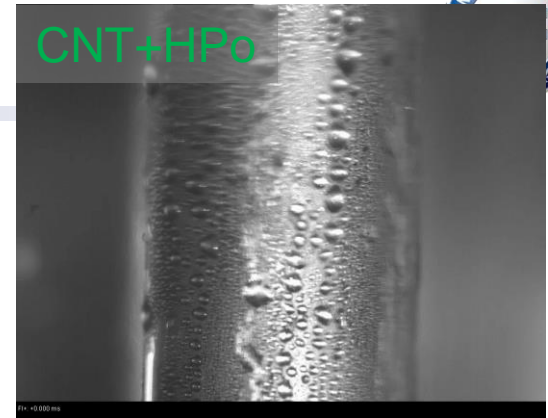
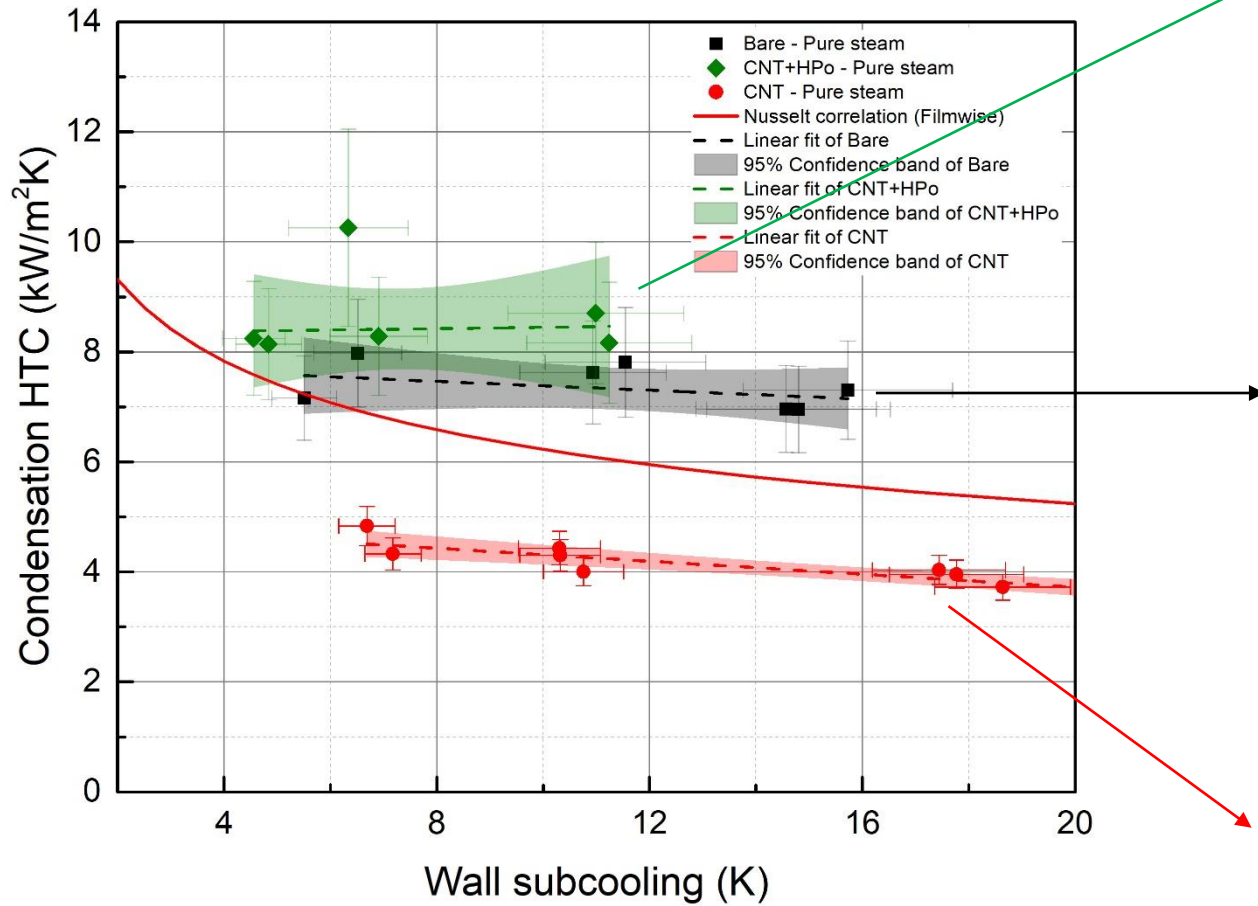
## Experimental set up (Test section)



Steam flow area	0.02 m <sup>2</sup>
Test specimen Length	1.18 m
Test specimen diameter	19.05 mm
Steam pressure	1 bar
Steam mass flux	0.1 kg/m <sup>2</sup> s ( ~ 0.16 m/s)
Coolant mass flux	1600 kg/m <sup>2</sup> s ( ~ 1.6 m/s)

# Result

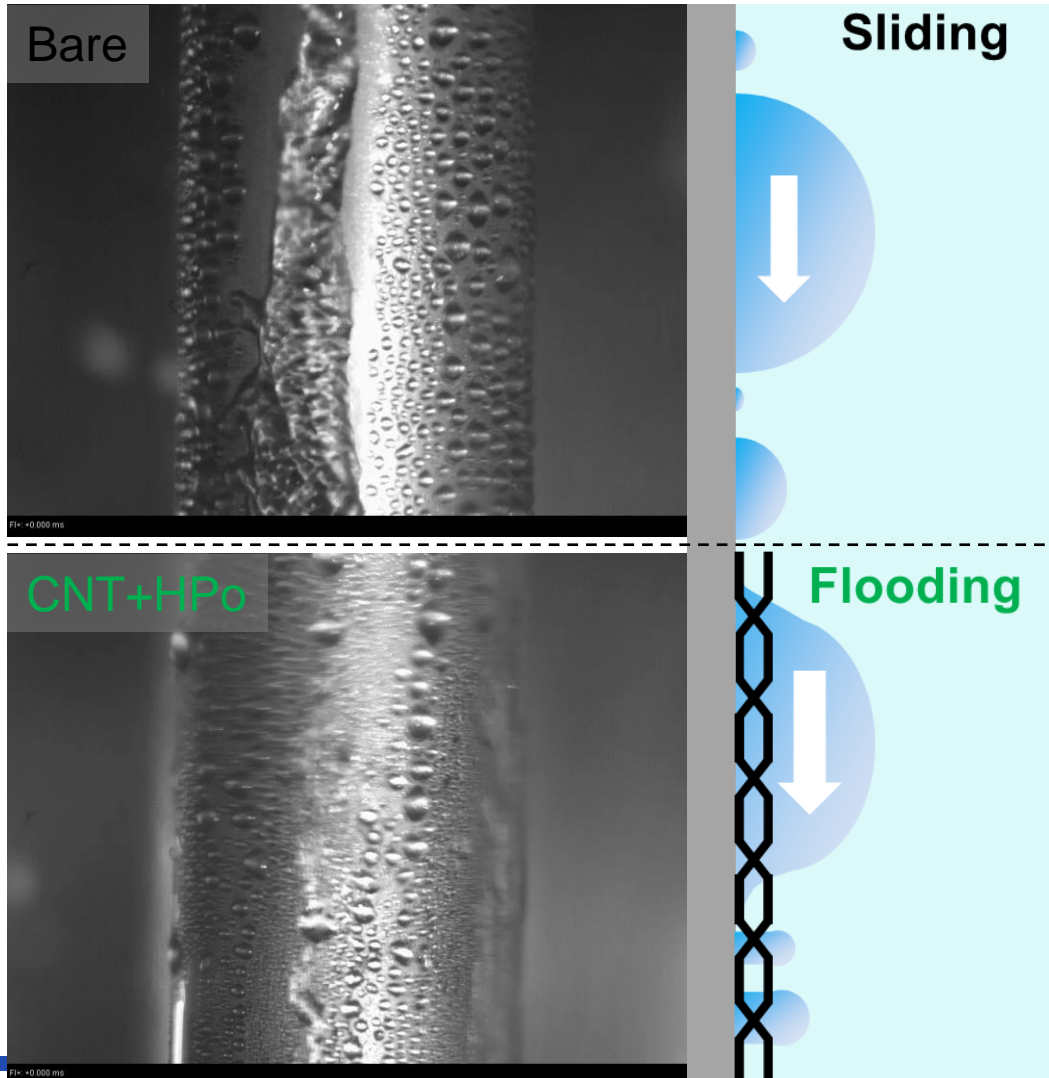
## Condensation heat transfer coefficient



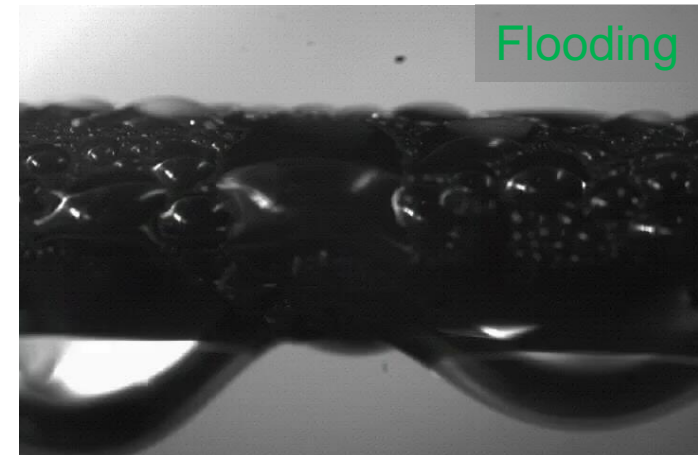
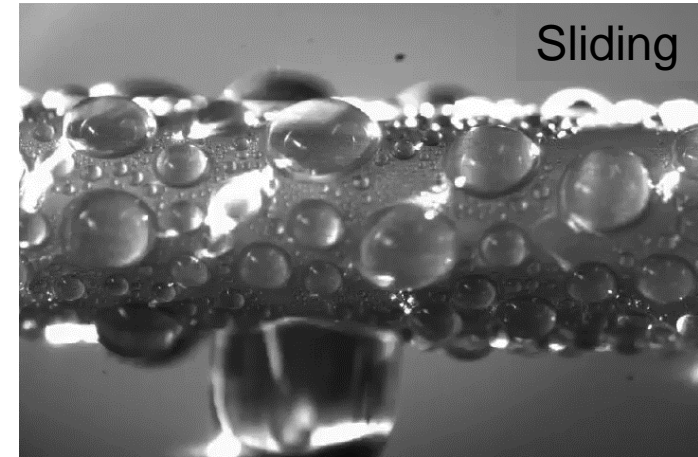
# Result



## Droplet morphology



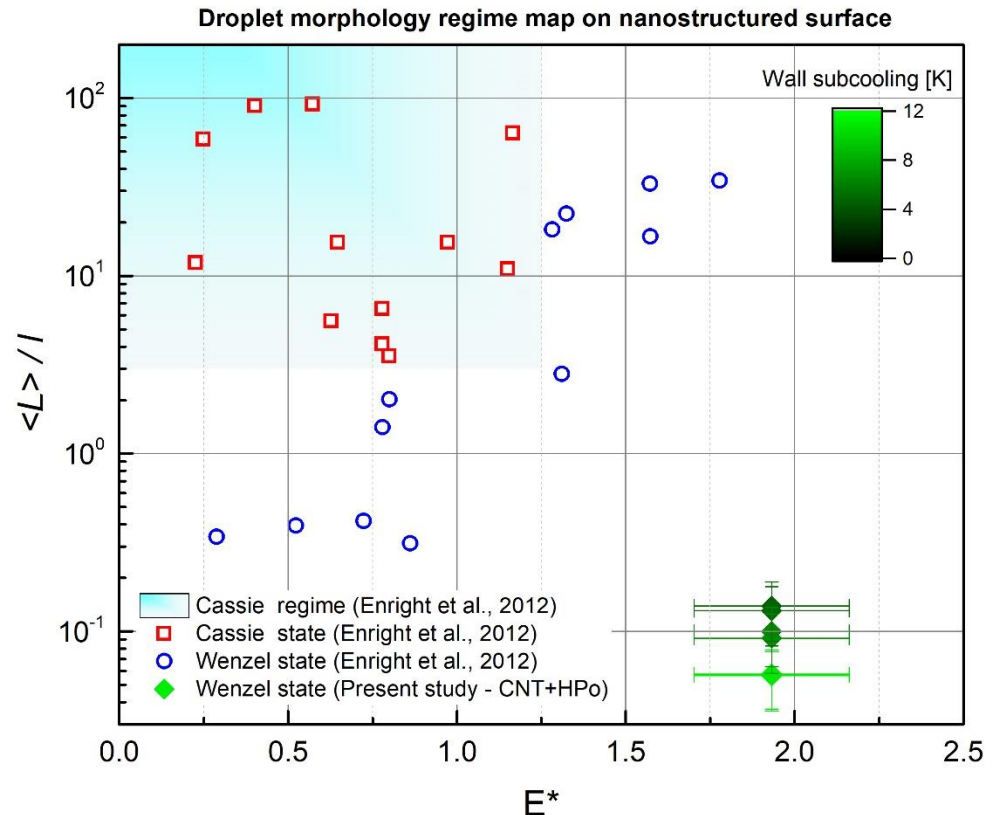
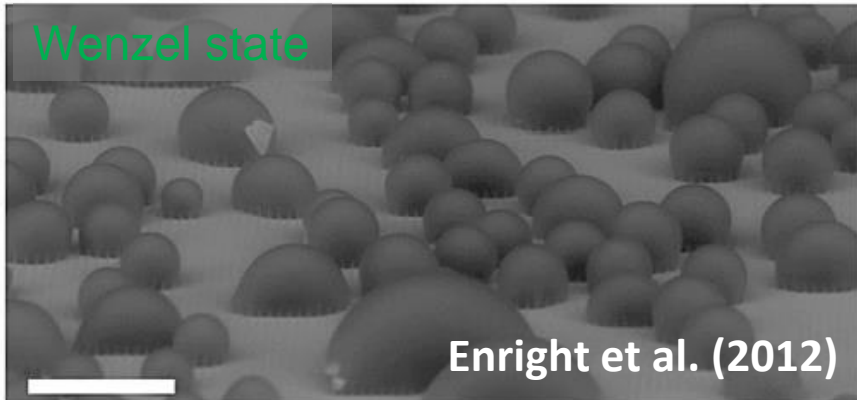
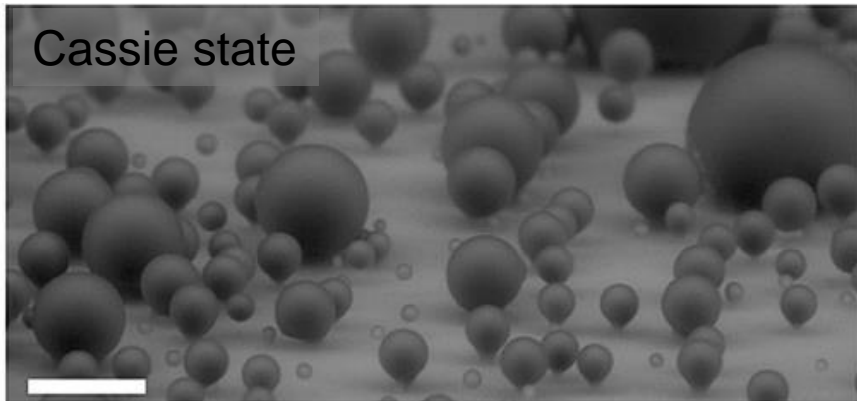
Miljkovic et al. (2013)



**AIHENA**

## Droplet morphology (Regime map)

- Droplet morphology regime map was developed by Enright et al.
- CNT+HPo surface was in Wenzel state region, and it caused flooding condensation

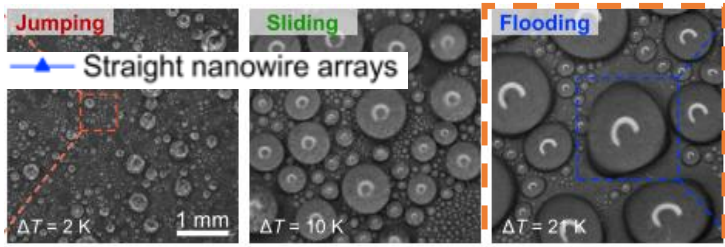
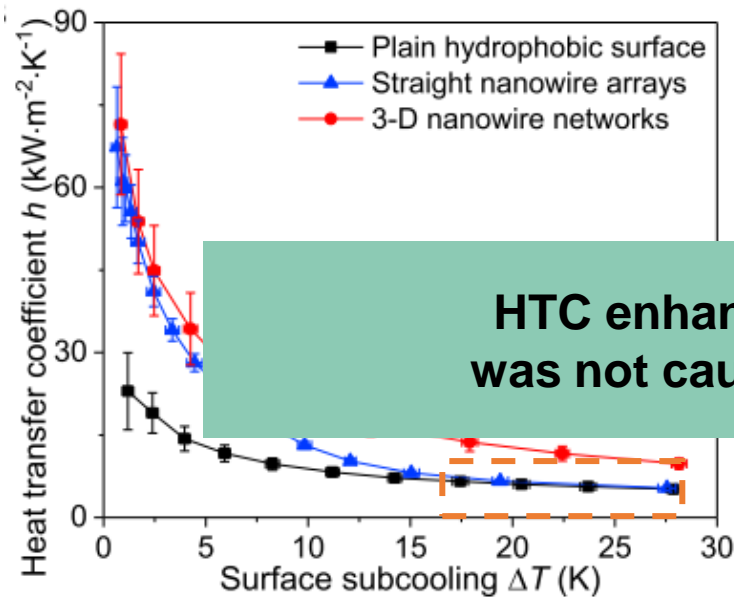


# Result

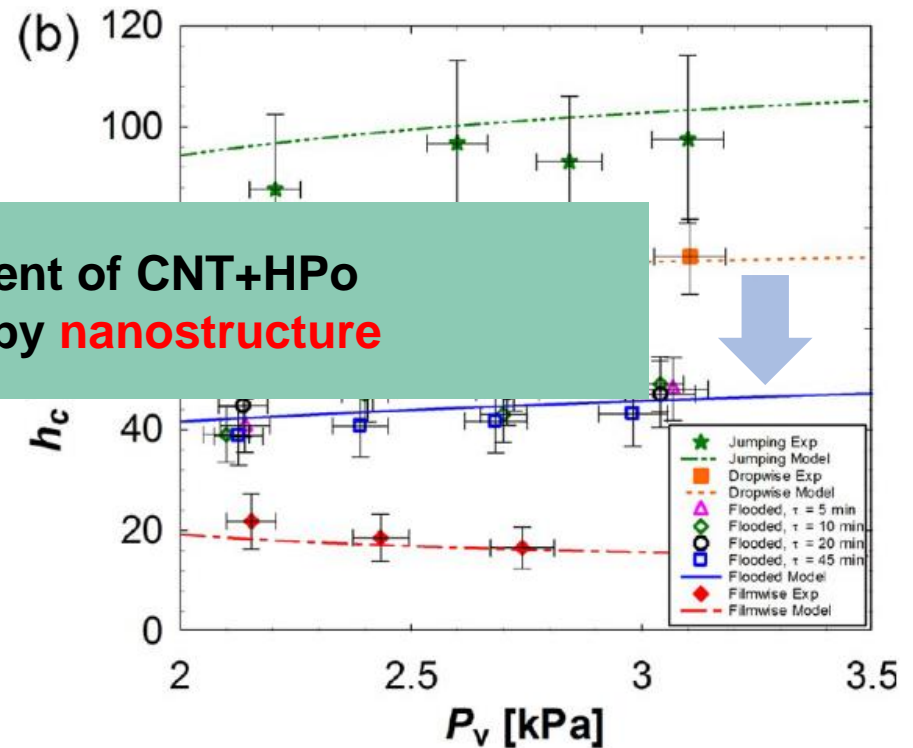
## Droplet morphology (Condensation heat transfer)

- **Flooding has no enhancement** of condensation heat transfer than the sliding

Wen et al. (2018)



Miljkovic et al. (2013)



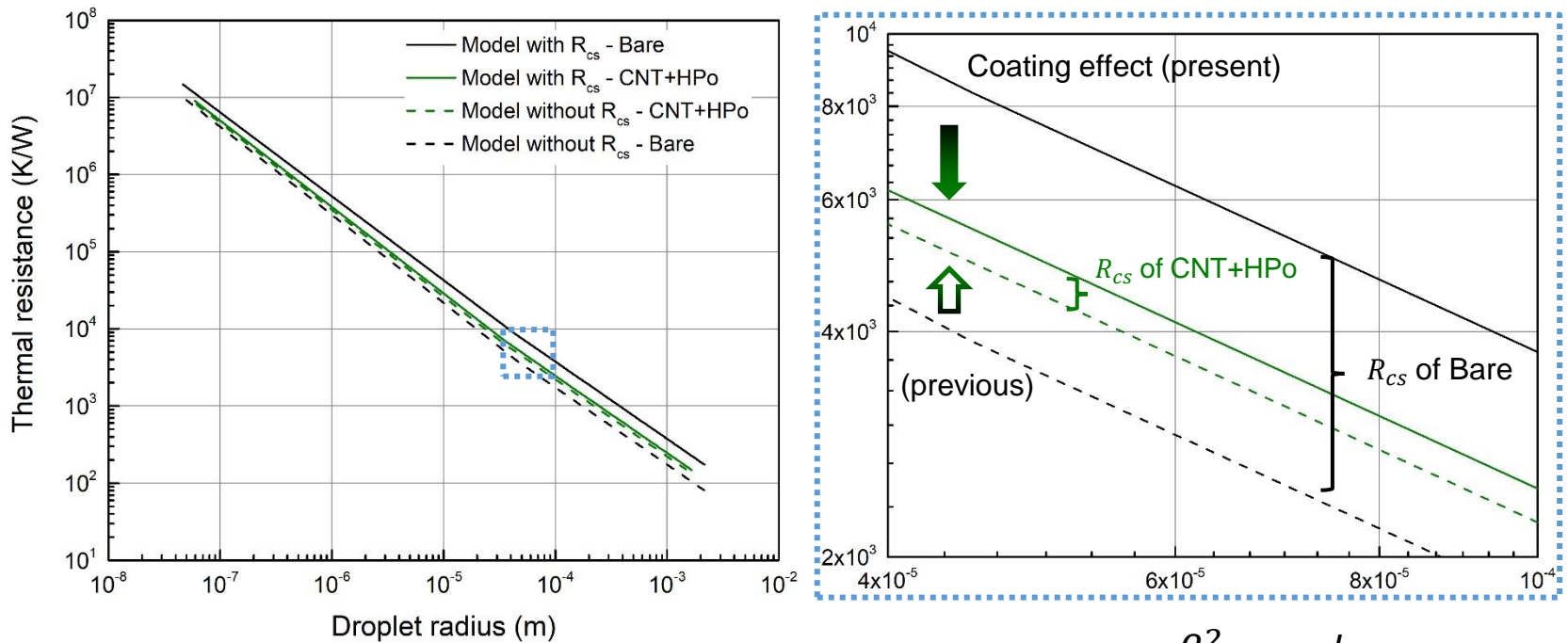


# Result

## Effect of the constriction resistance

- The CNT layers of the CNT+HPo surface can reduce the constriction resistance
- Constriction resistance was inserted to the condensation HTC model

### Thermal resistance of the single droplet



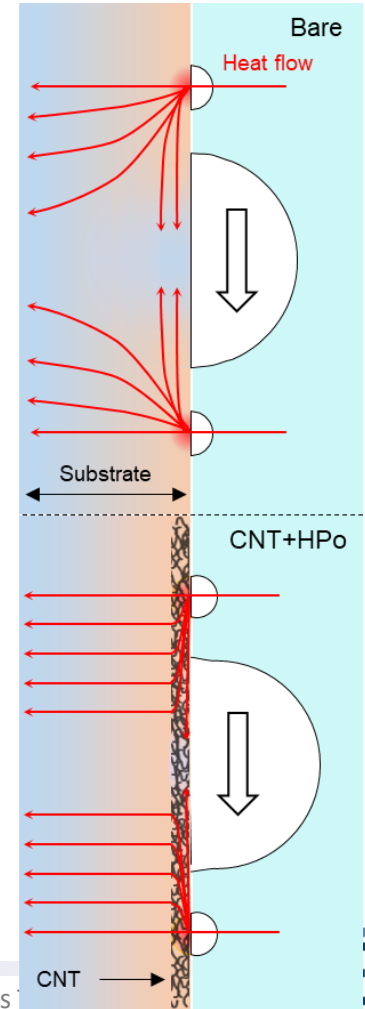
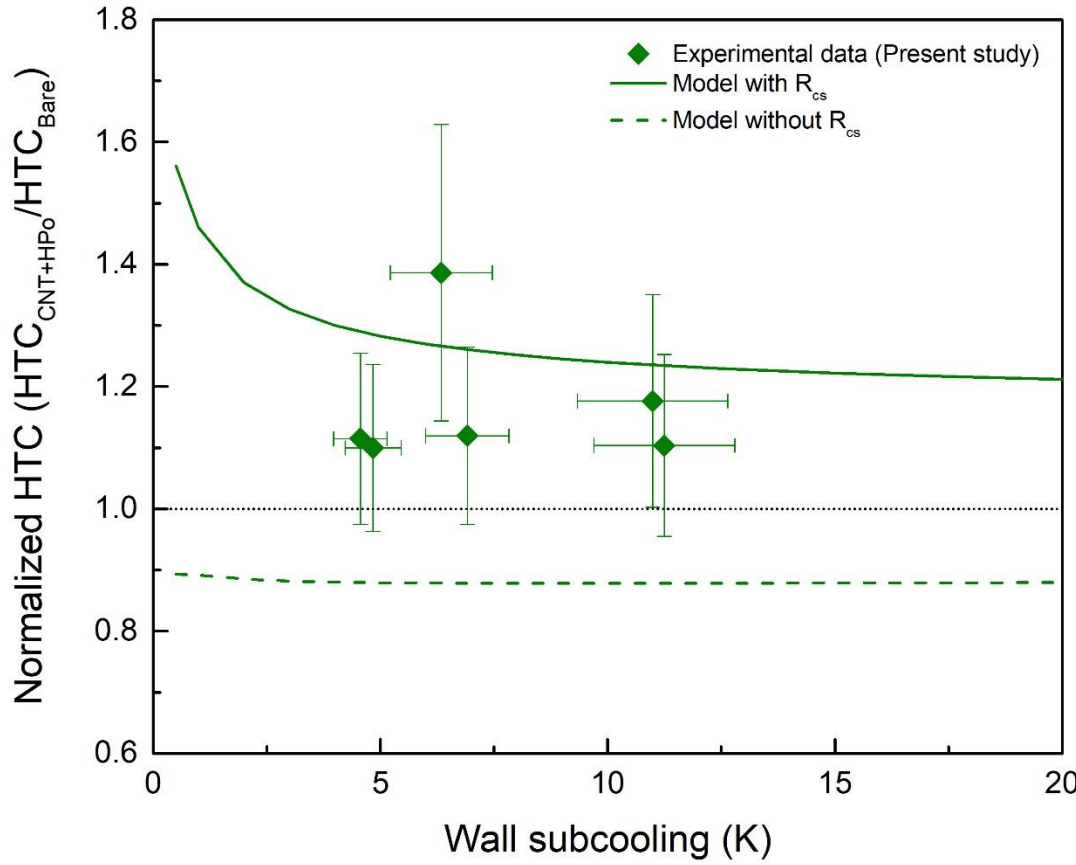
$$R = R_{lv} + R_{drop} + R_{coat} + R_{ls} + R_{cs}$$

$$R_{cs} \approx \frac{\beta^2}{(1 - \beta)^2} \frac{\psi}{4k_s r}$$

# Result

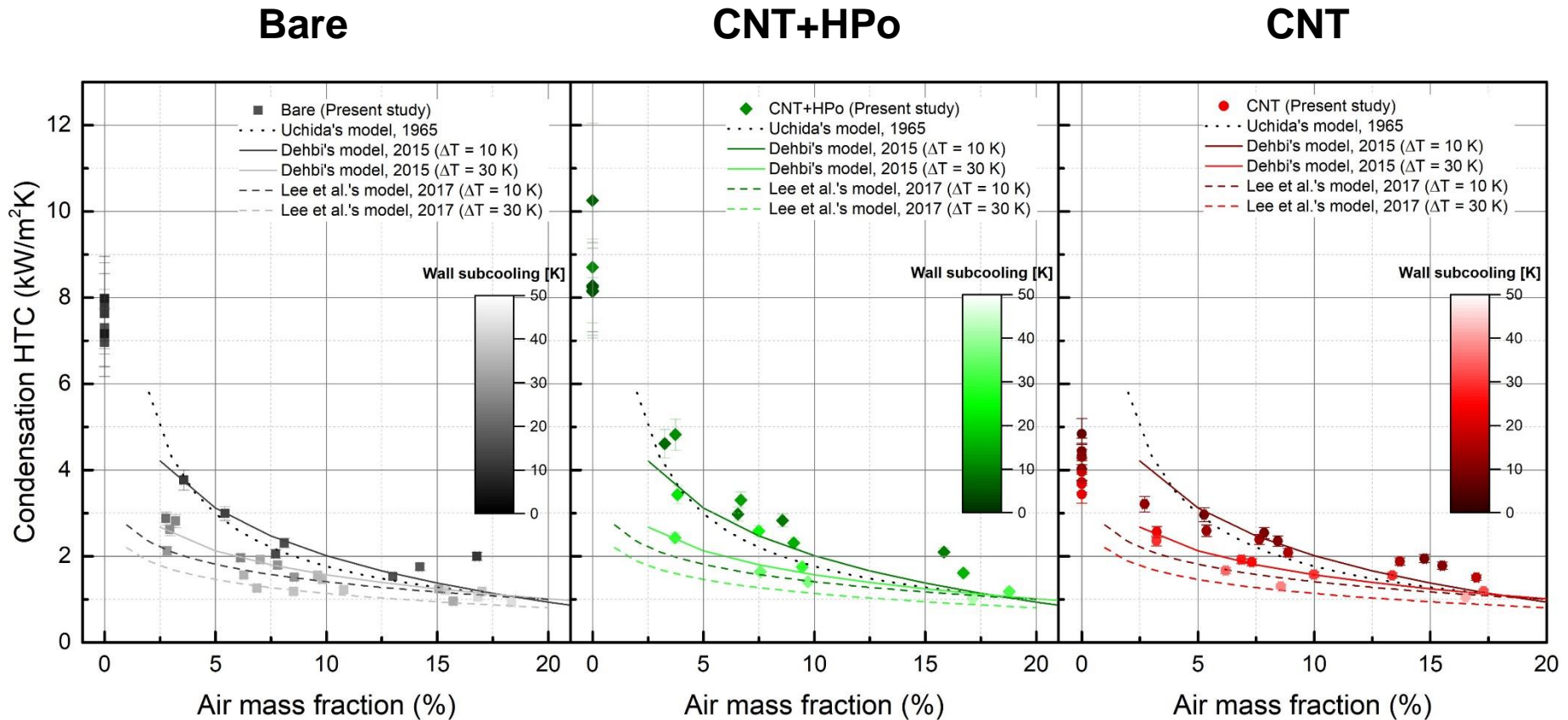
## Effect of the constriction resistance

- The present model (with constriction) is in line with the experimental result



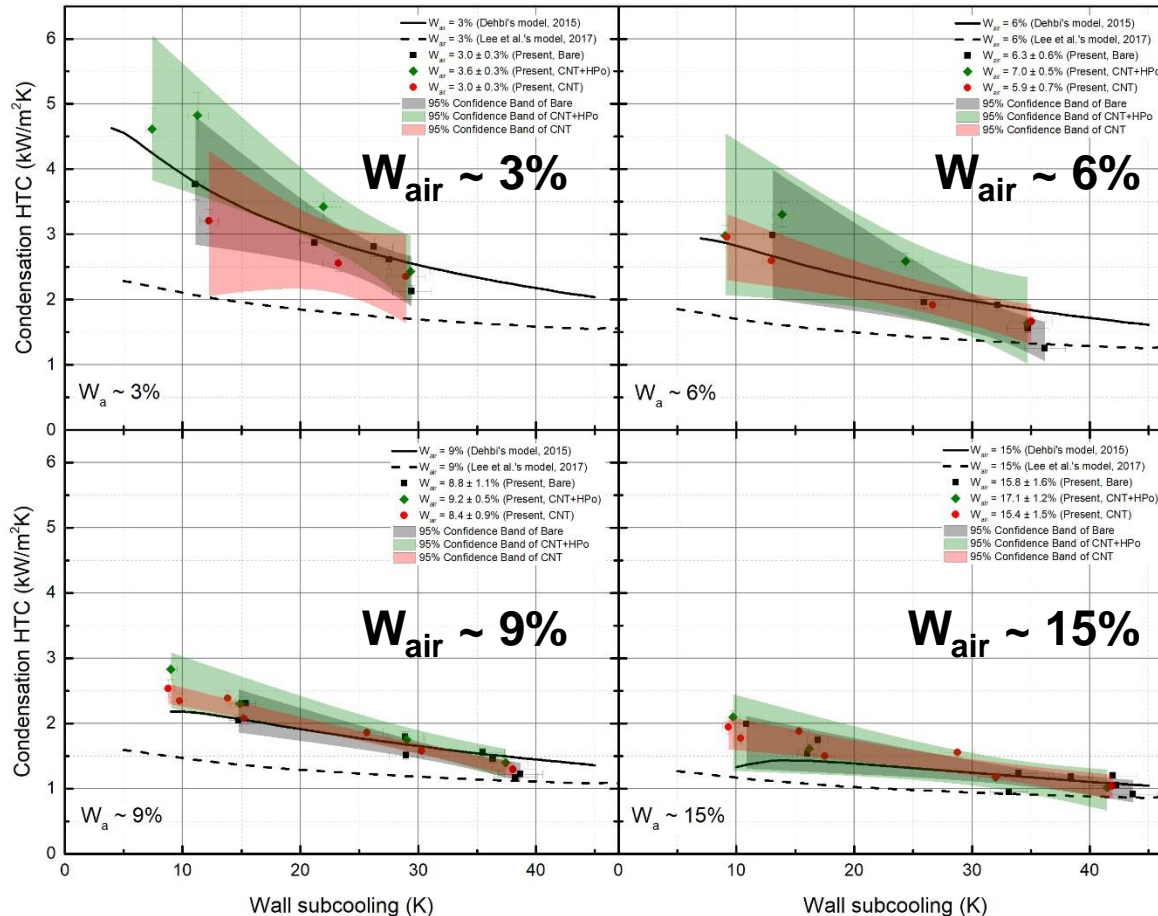
## Effect of the non-condensable gas

- The condensation HTC of each surface decreased as air mass fraction increased



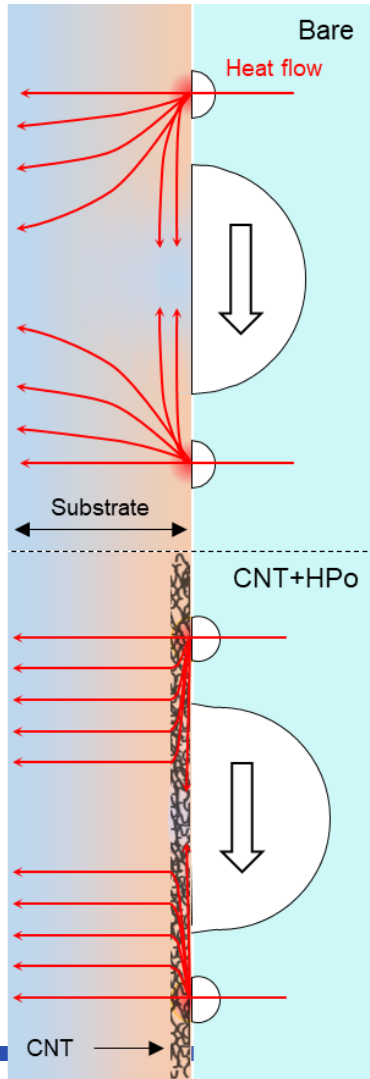
## Effect of the non-condensable gas

- Dropwise condensation HTC was similar with the filmwise as air fraction increased



# Conclusion

## Effect of CNT layers on the dropwise condensation



LbL assembled CNT coatings



Reduce the constriction resistance



Condensation HTC enhancement

# Acknowledgement



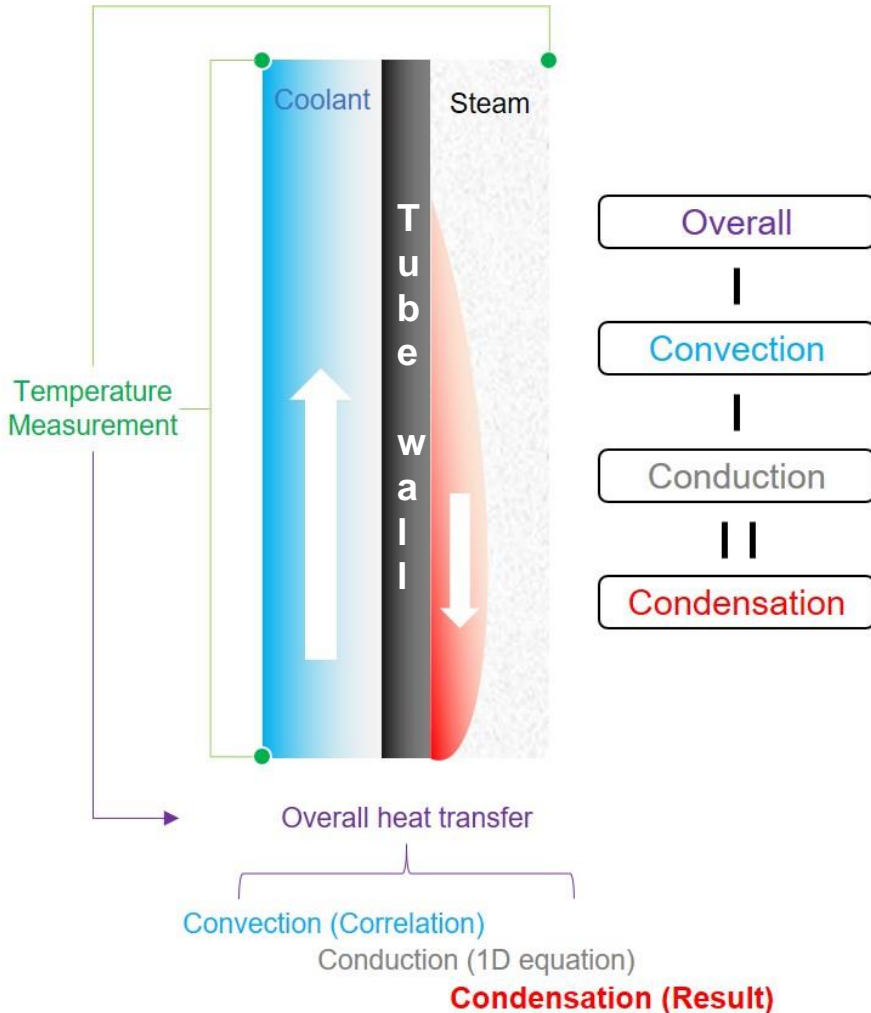
This work was supported by the National Research Foundation of Korea (NRF) funded by the Ministry of Science & ICT (MSIT, Korea) grant numbers [NRF-2016R1A5A101391921]



Thank you  
for your  
attention

# Appendix

## Data reduction (LMTD method)



$$U = \frac{Q}{A_{surface} \Delta T_{LMTD}} \left\{ \begin{array}{l} Q = \dot{m} C_p (T_{out} - T_{in}) \\ \Delta T_{LMTD} = \frac{T_{out} - T_{in}}{\ln \left( \frac{T_{steam} - T_{in}}{T_{steam} - T_{out}} \right)} \end{array} \right.$$

$$Nu = 0.023 Re^{0.8} Pr^{0.4}$$

$$R_{cond} = \frac{\ln(r_2/r_1)}{2\pi kL}$$

$$h = \frac{1}{\frac{1}{U} - \frac{d_o}{d_i} \frac{1}{h_{conv}} - \frac{d_o}{k} \ln \frac{d_o}{d_i}}$$

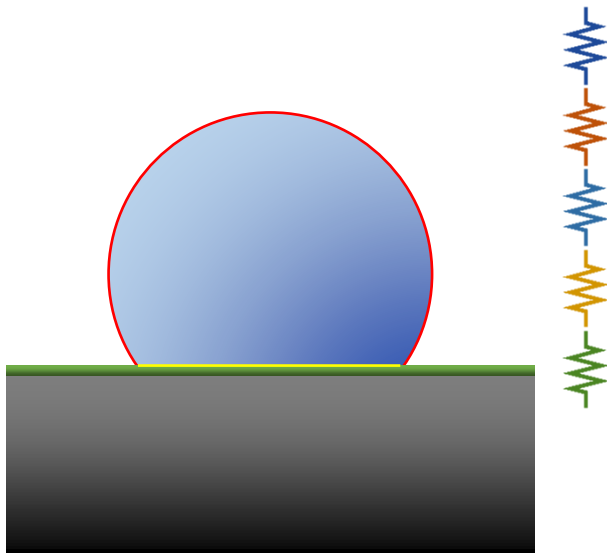
$$T_{sub} = \frac{Q}{h A_{surface}}$$



# Appendix

## Dropwise condensation HTC model

$$\Delta T = \Delta T_{curv} + \Delta T_{lv} + \Delta T_{drop} + \Delta T_{ls} + \Delta T_{coat} + q_d \times R_{cs}$$



$$\Delta T_{curv} = \frac{2T_{sat}\sigma}{h_{fg}\rho_l r}$$

$$\Delta T_{lv} = \frac{q_d}{2\pi r^2(1 - \cos \theta)h_{lv}}$$

$$\Delta T_{drop} = \frac{\theta q_d}{4\pi r k_l \sin \theta}$$

$$\Delta T_{ls} = \frac{q_d}{\pi r^2 \sin^2 \theta h_{ls}}$$

$$\Delta T_{coat} = \frac{\delta q_d}{\pi r^2 \sin^2 \theta k_{coat}}$$

$$\therefore q_d = \frac{\Delta T - \frac{2T_{sat}\sigma}{h_{fg}\rho_l r}}{\frac{\delta}{k_{coat} \sin^2 \theta} + \frac{r\theta}{4k_l \sin \theta} + \frac{1}{2h_{lv}(1 - \cos \theta)} + \frac{1}{h_{ls} \sin^2 \theta}}$$

## Dropwise condensation HTC model

$$n(r) = \frac{1}{3\pi r_e^3 r_{max}} \left( \frac{r_e}{r_{max}} \right)^{-\frac{2}{3}} \frac{r(r_e - r_{min})}{r - r_{min}} \frac{A_2 r + A_3}{A_2 r_e + A_3} \exp(B_1 + B_2)$$

$$A_1 = \frac{\Delta T}{2h_{fg}\rho_l}$$

$$A_2 = \pi r(1 - \cos \theta)(R_{drop} + R_{cs})$$

$$A_3 = \pi r^2(1 - \cos \theta)(R_{vl} + R_{ls} + R_{coat})$$

$$B_1 = \frac{A_2}{\tau A_1} \left[ \frac{r_e^2 - r^2}{2} + r_{min}(r_e - r) - r_{min}^2 \ln \frac{r - r_{min}}{r_e - r_{min}} \right]$$

$$B_2 = \frac{A_3}{\tau A_1} \left[ (r_e - r) - r_{min} \ln \frac{r - r_{min}}{r_e - r_{min}} \right]$$

$$\tau = \frac{3r_e^2(A_2 r_e + A_3)^2}{A_1(11A_2 r_e^2 - 14A_2 r_e r_{min} + 8A_3 r_e - 11A_3 r_{min})}$$

$$R_{cs} \approx \frac{\beta^2}{(1 - \beta)^2} \frac{\psi}{4k_s r}$$

$$\beta = \frac{r^2}{b^2}$$

$$\psi = \left[ \frac{32}{3\pi^2} \left( 1 - \frac{r}{b} \right)^{1.5} \right]$$

## Dropwise condensation HTC model

$$N(r) = \frac{1}{3\pi r^2 r_{max}} \left( \frac{r}{r_{max}} \right)^{-\frac{2}{3}}$$

$$r_{max} = \left( \frac{6(\cos \theta_r - \cos \theta_a) \sin \theta \sigma_{lv}}{\pi(2 - 3 \cos \theta + \cos^3 \theta) \rho_l g} \right)^{0.5}$$

$$r_e = (4N_s)^{-1/2}$$

$$N_s = \frac{0.037}{r_{min}^2}$$

$$q''_{drop} = \left( \int_{r_{min}}^{r_e} q_{drop}(r) n(r) dr + \int_{r_e}^{r_{max}} q_{drop}(r) N(r) dr \right)$$

## Dropwise condensation HTC model

$$\Delta\Psi(r) = \int [\Delta g + (p_{sat}(T_{sat}) - p_l)v_l dm_l + \sigma_{lv}A_{lv} + \sigma_{sl}A_{sl} - \sigma_{sv}A_{sv}]$$

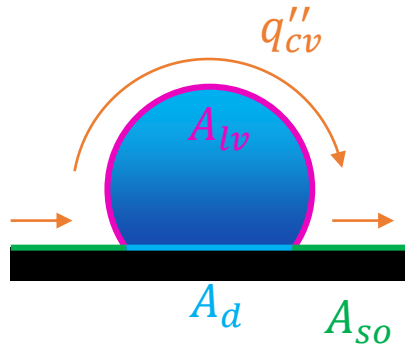
$$\Delta g = \frac{h_{fg}(T_l - T_{sat})}{T_{sat}} + v_l(p_l - p_{sat}(T_{sat}))$$

$$dv = A_s d\bar{\varepsilon} = \pi r^3 \sin^3 \theta \frac{(1 - \cos \phi)^2}{\sin^4 \phi} d\phi$$

$$\therefore \Delta\Psi = \rho_l \pi r^3 \sin^3 \theta \int_0^\theta \left( \frac{h_{fg}}{T_{sat}} \frac{(1 - \cos \phi)^2}{\sin^4 \phi} \times \left( -\Delta T_{sub} + \frac{q_{drop}(r)\delta}{\pi r^2 \sin^2 \theta k_{coat}} + \frac{q_{drop}(r)\phi}{4\pi r \sin \theta k_l} + \frac{q_{drop}(r)}{\pi r^2 \sin^2 \theta h_{ls}} \right) + \sigma_{lv}(2 - 3 \cos \theta + \cos^3 \theta)\pi r^2 \right) d\phi$$

$$\left. \frac{\partial \Delta\Psi}{\partial r} \right|_{r=r_{min}} = 0$$

## Dropwise condensation HTC model



$$Nu = \frac{h_F l}{k_v} = 0.664 Re^{1/2} Pr^{1/3} \quad Re < 5 \times 10^5$$

$$Nu = (0.037 Re^{0.8} - 871) Pr^{1/3}, Re > 5 \times 10^5$$

$$A_d = \left( \int_{r_{min}}^{r_e} \pi r^2 \sin^2 \theta n(r) dr + \int_{r_e}^{r_{max}} \pi r^2 \sin^2 \theta N(r) dr \right) A$$

$$A_{so} = A - A_d$$

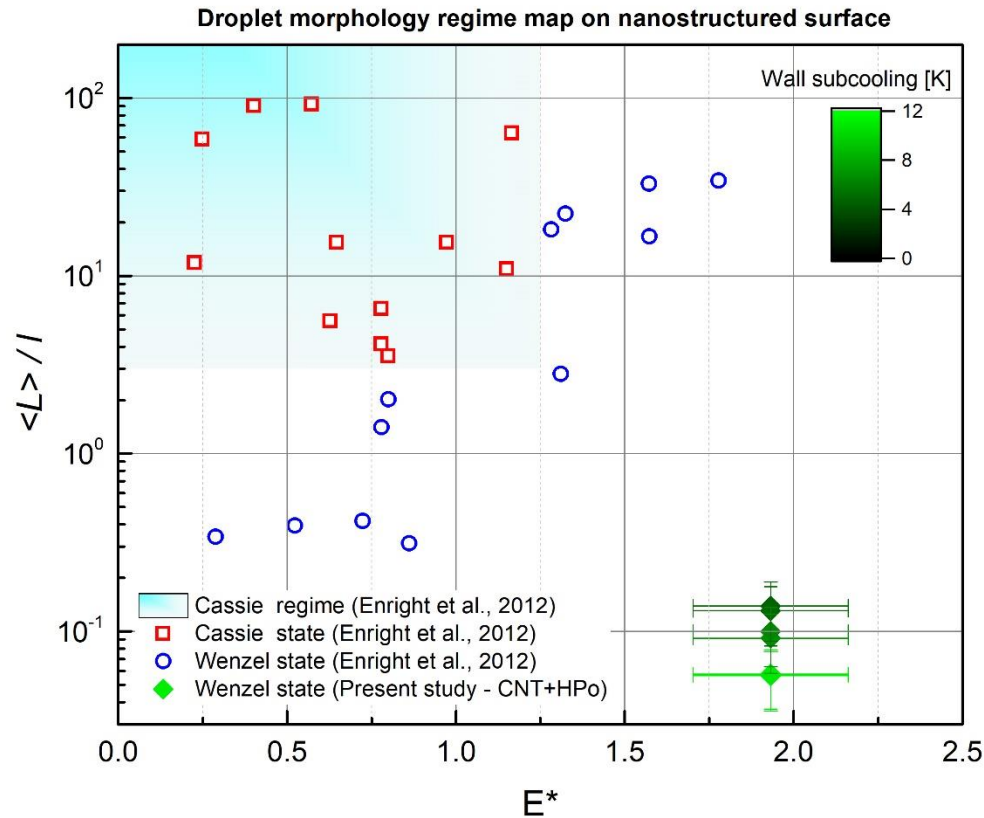
$$A_{lv} = \left( \int_{r_{min}}^{r_e} 2\pi r^2 (1 - \cos \theta) n(r) dr + \int_{r_e}^{r_{max}} 2\pi r^2 (1 - \cos \theta) N(r) dr \right) A$$

$$q''_{cv} = (A_{so} + A_{lv}) h_F \Delta T / A$$

$$\therefore q'' = q''_{cv} + q''_{drop}$$

## Droplet morphology regime map

$$\langle L \rangle = 2 \times r_e$$



$$E^* = -1/r \cos \theta_a$$