

A Pressure Drop Model for Duct Type Spacer Grid

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105

305-600

Abstract

A multiple parallel flow path model and pressure loss model of the each component are proposed to predict the pressure loss coefficient of the duct type grid in single-phase flow. The multiple parallel flow path model is generated by the energy conservation and momentum equations. The channel specific component pressure loss coefficients are combined on the basis of the free flow principle. The available literatures provide pressure loss coefficient needed for each component in a channel. The proposed model reasonably predicts available grid pressure loss data, therefore the model is suitable as a tool for the grid optimization in grid development phase.

1.

가
가

가 . 가
 가 . 가

PWR 1
 가 :
 1) 가
 2) 가

1) PWR BWR
 2) 80

$$\Delta P = \Delta P_o + n\Delta P_{sg} \quad (1)$$

(2)

$$C = \frac{\Delta P_{sg}}{\frac{1}{2} \rho V_o^2} \quad (2)$$

ΔP_{sg} ρ , V_o

Rehme(1973) FBR

Reynolds 가 10^5
 6 7

Cevolani(1995)

Reynolds

Rehme

Kim(1993)

가

FBR

Kim

FBR

Oh(1998)

2.

(I) (II), (III, IV), (V, VI),
 (VII)

2.1

3 b c 3 가
 - 3 a d
 - 3 a c, d P
 가 3 (3)

$$P_a + \frac{\rho}{2} V_a^2 = P_c + \frac{\rho}{2} V_{c,i}^2 + K_{ac,i}^b \frac{\rho}{2} V_{b,i}^2 ; i = 1, K, N \quad (3)$$

$K_{ac,i}^b$ a c b
 (stagnation loss coefficient)

가 c $V_{c,i} = V_{b,i} \frac{A_{b,i}}{A_{c,i}}$

(3) (4)

$$P_a + \frac{\rho}{2} V_a^2 - P_c = \frac{\rho}{2} K_{ac,i}^b V_{b,i}^2 + \frac{\rho}{2} \left(\frac{A_{b,i}}{A_{c,i}} \right)^2 V_{b,i}^2 ; i = 1, K, N \quad (4)$$

a b (5-a) (5-b) (5-c)

$$A_b = \sum_{i=1}^N A_{b,i} \quad (5-a) \quad A_c = \sum_{i=1}^N A_{c,i} \quad (5-b) \quad V_b = V_a \frac{A_a}{A_b} \quad (5-c)$$

a b

$$A_a V_a = \sum_{i=1}^N A_{b,i} V_{b,i} \quad (6)$$

(6) (5-c) (7)

$$1 = \sum_{i=1}^N \frac{A_{b,i} V_{b,i}}{A_b V_b} \quad (7)$$

a c

$$P_a - P_c = C_{ac}^b \frac{1}{2} V_b^2 \quad (8)$$

(4) (7), (8) a c

$$C_{ac}^b = \frac{1}{2} \frac{A_b V_b^2}{A_a V_a^2} \left[\sum_{i=1}^N \frac{A_{b,i} V_{b,i}^2}{A_b V_b^2} + \frac{A_{b,i} V_{b,i}^2}{A_{c,i} V_{c,i}^2} \right] \quad (9)$$

가 c d (10)

$$P_c - P_d = \frac{r}{A_d} \sum_{i=1}^N A_{c,i} V_{c,i} (V_d - V_{c,i}) \quad (10)$$

(8) c d (10)

c d

$$C_{cd}^b = 2 \frac{A_b V_b^2}{A_d V_d^2} \left[\frac{A_b}{A_d} \sum_{i=1}^N \frac{A_{b,i} V_{b,i}^2}{A_b V_b^2} + \frac{A_{b,i} V_{b,i}^2}{A_{c,i} V_{c,i}^2} \right] \quad (11)$$

(9) (11) a c C_{ac}^b c d

$$C_{cd}^b = A_b$$

a d

$$C_{ad}^b = C_{ac}^b + C_{cd}^b \quad (12)$$

2.2

(free flow principle) (13) [6].

$$K_{ac}^b = K_f + K_{un} + K_{ai} + K_{ad} + K_{dh} + K_{ic} + K_{ah} + K_{ah} + K_{ah} + K_{db} + K_{df} \quad (13)$$

1 [6,7].

(K_f) :
 가
 Blasius
 [5]

(K_{un}) , (K_{ai}) , (K_{ad}) , (K_{ah}) :
 Reynolds

3
 2 (wake)
 2

(K_{dn}) , (K_{ic}) :

가 K_{cd}^b

(K_{ah}) :
 (sharp edged orifice)
 가

(K_{db}) :
 가 가

(K_{df}) :

0.5 가

3.

가
 Type A 17X17 Westinghouse 2
 Type B 14X14 ABB-CE
 4
 Type A 1.4% 8.4% Type B
 Reynolds Reynolds
 Reynolds Reynolds 가
 가

Table 2. Geometric Parameters of the Duct Type Spacer Grids

Parameter	Type A	Type B
Rod Array	17×17	14×14
Rod Diameter, D[mm]	9.55	11.2
Pitch to Diameter, P/D	1.319	1.316
Grid Height, L[mm]	44.45	44.45
Strap Thickness, t[mm]	0.381	0.356
Bending Angle, δ [deg]	20	20
Number of Guide Tubes	25	5

4.

(free flow principle)

Westinghouse ABB-CE

Reynolds

가

가

NOMENCLATURES

SUBSCRIPT

BWR	boiling water reactor	
PWR	pressurized water reactor	
FRB	fast breeder reactor	
A	flow area	[m ²]
AP	projected area	[m ²]
C	static pressure loss coefficient	
Dh	hydraulic diameter	[m]
K	stagnation pressure loss coefficient	
L	friction length	[m]
P	static pressure	[Pa]
PW	wetted perimeter	[m]
R	radius of curvature	[m]
Re	Reynolds number	
n	number of grids in a bundle	

ad	area decrease
ah	inlet anti hanging tab
aht	exit anti hanging tab
ai	area increase
db	duct bending
df	diffuser
dn	downstream nugget
f	friction
ic	inlet contraction
l	local
m	grid middle level
o	bare rod
r	reference (b position in Fig. 3)
rd	rod
sd	sudden decrease
sg	spacer grid
t	tab
un	upstream nugget

GREEK LETTERS

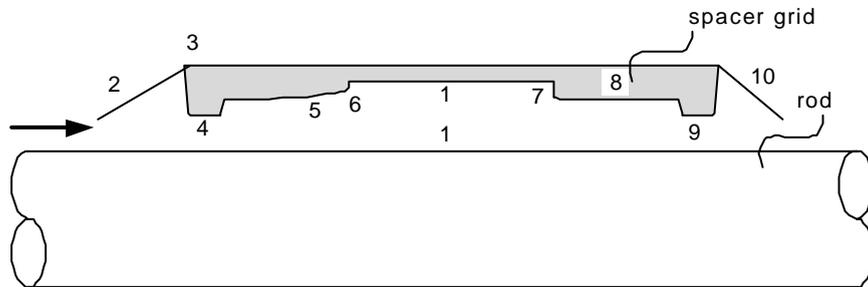
Δ	difference	
α	duct width	[m]
β	duct length	[m]
ρ	density	[kg/m ³]
δ	bending angle	
e	area blocking ratio	

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Table 1. Pressure Loss Coefficient Model for Each Component

No	Component	Model	Ref.
1	Friction, K_f	$\frac{f L}{4A_l} \left(\frac{A_r}{A_l} \right)^{1.8} \left(\frac{Dh_o}{Dh_l} \right)^{-0.2} + 0.204 PW_{rd} + 0.184 PW_{st}$	[5]
2	Inlet anti tab, K_{ah}	$1.707 \cdot \left(\frac{A_l}{A_o} \right)^{0.375} + \left(\frac{A_l}{A_r} \right)^2 \left(\frac{A_r}{A_l} \right)^2$, $A_l = A_r - AP_t$	[7] diag. 4-11
3	Inlet contraction, K_{ic}	$0.5 \cdot \left(\frac{A_r}{A_o} \right)^{3/4}$	[7] diag. 4-9
4	Upstream nugget, K_{un}	$0.2 \cdot \frac{e_{ng}}{1 - e_{ng}}$, $e_{ng} = \frac{AP_{ng}}{A_r}$	[6] ch. 3 fig. 21
5	Diffuser loss, K_{df}	$0.5 \cdot \left(\frac{A_r}{A_l} \right)^2$	[7] diag. 4-1
6	Sudden area increase, K_{ai}	$0.5 \cdot \frac{e_{ai}}{1 - e_{ai}}$, $e_{ai} = \frac{A_{sd}}{A_m}$	[6] ch. 3 fig. 22
7	Sudden area decrease, K_{ad}	$0.4 \cdot e_{ad}$, $e_{ad} = \frac{A_{sd}}{A_m}$	[6] ch. 3 fig. 22
8	Duct bending, K_{db}	$116 + 8.65 \left(\frac{Dh_r}{2R} \right)^{1.32} \left(\frac{b}{a} \right)^{0.34} e^{-0.25 \left(\frac{Dh_r}{Dh_o} \right)^{-0.25}} \cdot d \cdot \left(\frac{V_r}{V_o} \right)^{-0.25}$	[7] diag. 4-1
9	Downstream nugget, K_{dn}	$0.5 \cdot \left(\frac{A_l}{A_r} \right)^{3/4} \left(\frac{A_r}{A_l} \right)^2$, $A_l = A_r - AP_{ng}$	[7] diag. 4-9
10	Exit anti tab, K_{ah}	$1.7 \cdot \frac{e_{ah}}{1 - e_{ah}}$, $e_{ah} = \frac{AP_t}{A_r}$	[6] ch. 3 fig. 8



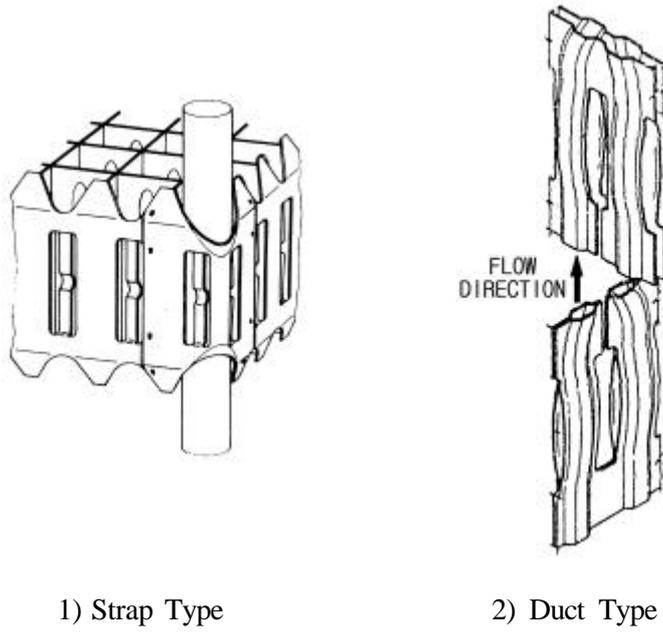


Figure 1. Spacer Grids

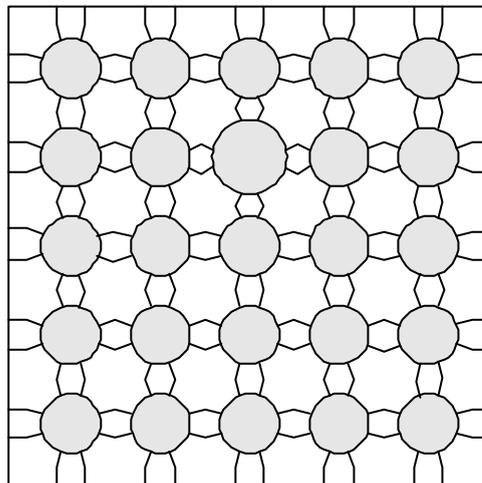


Figure 2. Subchannels for Duct Type Grid

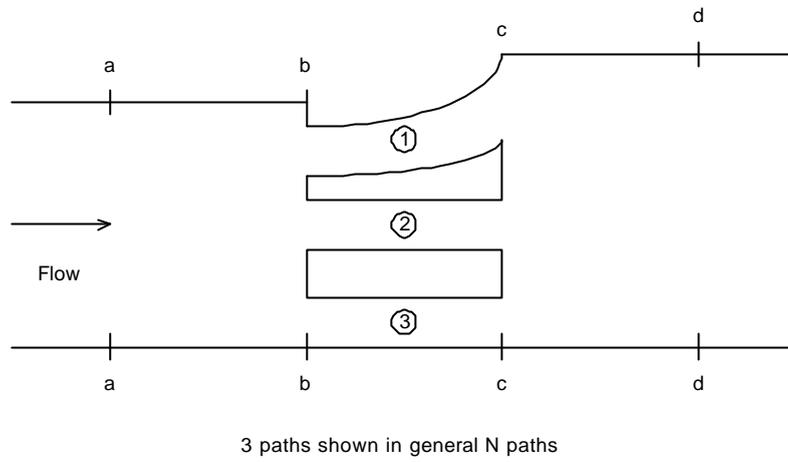


Figure 3. Multiple Parallel Flow Path

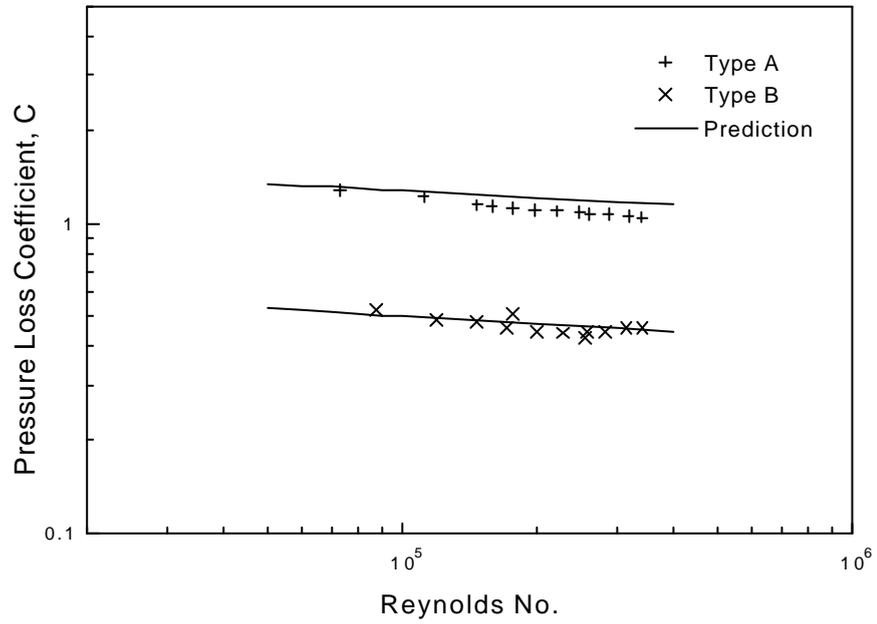


Figure 4. Evaluation of the Pressure Loss Model