A Study on CANDU Model Assessment of RELAP5/CANDU using RD-14M B9401 Multi-channel RIH Break Experiment

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ABSTRACT

B9401 experiment, performed in RD-14M[1] multi-channel facility, was analyzed using RELAP5/MOD3 and RELAP5/CANDU and compared with experiment results. The RELAP5/CANDU code has been developed since 1998, based on RELAP5, in order to have auditing tool of CANDU NPP. The RELAP5/CANDU code is under developing and they have not been assessed much for a CANDU reactor. Therefore, this study has been initiated with an aim to identify the code applicability in a CANDU reactor by simulating some of the tests performed in the RD-14M facility and to get the assessment results for RELAP5/CANDU code. The RD-14M test facility at Whiteshell Nuclear Research Establishment is a full-scale multi-channel pressurized-water loop. The RELAP5/MOD3 and RELAP5/CANDU analyses demonstrate the code's capability to predict reasonably the main phenomena occurred during the transient, in qualitative view. In quantitative view, the RELAP5/CANDU[4] predicted better than that of RELAP5. In the case of experiment that the stratification in fuel channel is dominant, it is expected that RELAP5/CANDU can give more accurate result than RELAP5.

I. INTRODUCTION

There are 34 CANDU reactors completed or under construction worldwide and in Korea, four CANDU have been operated in Wolsong site, named Wolsong unit 1,2,3 and 4. The safety of CANDU type nuclear power plant has been focused. In Canada, the effectiveness of emergency core cooling system (ECCS) and the core cooling in the absence of flow have been considered as "generic safety issues" identified by the Canadian regulatory body, AECB, as being applicable to all or most of the CANDU nuclear power plants in Canada.[9,10,11] To provide information on the effectiveness of ECCS in a CANDU reactor, various series of experiments has been carried out in the RD-14 pressurized water loop at the Whiteshell Nuclear Research Establishment from 1984 to 1987. As a following experimental facility, the RD-14M had been constructed and operated since 1988 for the multi channel behavior.

In previous study [3], several experiments performed in RD14 facility were analyzed using RELAP5/MOD3 and the applicability of RELAP5 in the area of CANDU analysis was demonstrates but some discrepancies also were observed. In critical header break analysis, there were some discrepancies on the header pressure prediction after ECI injection, sheath temperature behavior, and channel flowrate. The discrepancies were due to the complicated behavior of CANDU specific features and in order to analyze the CANDU nuclear power plant for auditing purpose, the CANDU specific models were needed. The RELAP5/CANDU[4] code has been developed since 1998 to have independent audit tool and now still under development. This code has been modified in the area of CANDU specific phenomena based on RELAP5/MOD3.2 gamma version. The important models were selected among the modified or new model using mathematics and engineering judgment based on experimental phenomena. However, it has not been fully assessed for the CANDU reactor.

In this study, the multi channel experiment B9401 was analyzed using RELAP5/MOD3.2 [6] and RELAP5/CANDU and compared to experiment results. The present study aims to assess RELAP5/MOD3.2 gamma version with RD-14 B9401 experiment, compare the result with that of RELAP5/CANDU in order to check the consistency between two codes, and validate the developing code.

II. Background

A. RD-14M Facility Description [1]

RD-14 was designed and constructed starting 1981. The RD-14 reference design chosen was two, 5.5 MW, 2-pass with 37-element single channel, (i.e., one channel per pass), with 1:1 scaling of vertical distances throughout the loop. This determined the sizing of piping and various components (e.g., steam generators, pumps, headers). The values for various loop parameters dictated by the choice of reference design were 5.5 MW maximum thermal power per pass, 590 kW/m maximum surface heat flux per pass and 24 kg/sec rated flow rate (one 37-element channel).

Table 1 Comparison of Characteristics of RD-14 and CANDO feactor				
Parameters	RD-14	RD-14M	Typical Reactor	
Operating Pressure (MPa)	10	10	10	
Loop Volume (m ³)	0.95	1.01	60.	
Heated Sections:	37-rod bundles	7-rod bundles	37-element bundle	
Number per pass	1	5	95	
Length (m)	6	6	12 x 0.5	
Rod diameter (mm)	13.1	13.1	13.1	
Flow tube Dia. (mm)	103.4	44.8	103.4	
Power (kW/channel)	5500.	3x750, 2x950 per pass	5410.	
Pumps:	single stage	same as RD-14	same as RD-14	
Impeller diameter(mm)	381	381	813	
Rated flow (kg/s)	24.	24.	24. (max/channel)	
Rated head (m)	224.	224.	215.	
Specific speed	565.	565.	2000	
Steam Generators:	recirculating U-tube	same as RD-14	same as RD-14	
Number of tubes	44	44	37/channel	
Tube diameter I.D.(mm)	13.6	13.6	14.8	
Secondary heat-	41	41	32.9/channel	
transfer area (m ²)				
Secondary Volume (m ³)	0.9	0.9	0.13	
Heated Section-to-Boiler	21.9	21.9	21.9	

Table 1 Comparison of Characteristics of RD-14 and CANDU reactor

The modification of RD-14 to RD-14M provides for the study of the interaction of multiple heated channels in parallel in a full height loop. As multiple channels, five 7-element heated sections per pass were chosen to replace the single, 37-element channel. The cross sectional area of the associated below header pipe-work was scaled at 7:37 to preserve heat and mass fluxes in the multi-channel facility.

As noted in reference [1], the large number of non-dimensional groups to be considered precludes the scaling of two-phase flow dynamics with complete similarity. However, if the model is made of a similar solid material and has a similar fluid under the same system pressures as the prototype, scaling is simplified. Reference [1] presents an appropriate set of similarity criteria to be used under such conditions. Using 1:1 scaling of vertical elevations and axial lengths simplifies the scaling of the facility. It is appropriate to choose the piping diameters such that the flow velocities will be scaled 1:1. This ensures that the characteristic transit times will be approximately equal in both the facility and the reactor.

In RD-14M, consideration was given to the several experimental program in the design of the loop, the loop peripherals and the loop instrumentation. The experimental programs were categorized into three groups, safety-type transients, process dynamics and control-type transients, and component-type transients.

B. Description of Experiment [1]

Top Elev. Difference (m)

A series of experiments to investigate the thermal-hydraulic responses of critical break with emergency coolant injection were progressed in the RD-14M test facility. The experiment used in this study is B9401 experiment – 30 mm inlet header break experiment with high pressure pumped emergency coolant injection.

The nominal initial conditions for the first experiment in this series, B9401, were 10.0Mpa(g) outlet header pressure, 4.0MW per pass nominal input power, 4.4 Mpa(g) steam pressure, and

186°C feed water temperature. Before the experiment, the loop was evacuated, filled and degassed, all instrument lines were vented, and instrument readings were checked and adjusted. The loop was warmed using low power and reduced pump speed. Input power and pump speed were then increased to bring the loop to the desired steady-state single phase starting conditions. The detailed sequence of events during the experiment was described in Table II.

A programmable pump-speed controller was used in this experiment to simulate pump rundown following a loss of class-IV power. The pump began ramping down at 12s. Cold water was injected into the loop when the primary pressure fell to or below the emergency coolant

Table 2. B9401 test (30mm RIH) Procedure

B9401	RELAP5	Event Description		
Time	Time			
0	0.	start data gathering		
10	10.	open break valve, p14 start		
12	12.	step input power to decay level		
		and RCP ramped down		
20.6	23.5	ECI isolation valve open		
22.8	22.8	Pressurizer tank (surge tank)		
116.2	116.2	isolated		
231	231.	HP ECI terminated, LP ECI start		
350.7	350.	Primary pumps off		
400.0	400.0	LP ECI terminated		
		End		

injection (ECI) pressure. The isolation valves at the ECI pipes to all four headers were opened as soon as the pressure in header 7 fell below 5.5 MPa. As long as the pressure in any header was above 5.5 MPa (pressure in the ECI tank), no ECI water entered that header. When the pressure in any header was below 5.5 MPa, ECI water entered the header at a rate determined by the pressure difference between the ECI tank and the header. Orifices in the ECI injection lines provide scaled simulation of reactor injection flow rate. This experiment was focused on multi

channel behavior such as sequential reflooding, unbalanced ECI, etc and this will described in the result.

C. RELAP5/CANDU code description [4]

As described in the above, the safety of CANDU plant has been focused due to increasing the number of CANDU plants. Until now, Korea had no independent audit calculation tool because CANDU plant has its own special design such as horizontal core, channel type core, header design, etc. Therefore, the development of RELAP5/CANDU code has been initiated by KINS cooperated with KAERI. The modifications were performed as following procedure;

- 1) RELAP5/MOD3.2 gamma version was selected as base code.
- 2) Identify important process and phenomena in CANDU
- 3) Prioritization of the selected process and phenomena using engineering judgment
- 4) The selected and prioritized items were divided into two groups, which were LOCA and non-LOCA, and perform the modification.

Until now, the modified and added models for RELAP5/CANDU as follows;

- 1) Critical Flow Model
- 2) Nuclear Kinetics Model
- 3) Critical Heat Flux Model
- 4) Reactor Core Control Model
- 5) Valve and Spray Model
- 6) Improvement of Horizontal Flow Regime Map
- 7) Heat Transfer Model in Horizontal Channel

Details are described in reference xx.

III. RELAP5 SYSTEM MODEL

System model for RELAP5 calculation is shown in Figure 1 and 2, which are basically similar ones found in CATHENA model [6,7,8] and therefore may help reduce the effect of nodalization. The system model composes of primary heat transport system including heaters and pumps, secondary system, ECI system, accumulator, and break model. Especially, the ECI pipings were modeled in order to simulate the ECI flow-splitting behavior. The same nodalization was also used to RELAP5/CANDU analysis.

IV. ANALYSIS RESULTS AND COMPARISONS

A. RELAP5 Results

Header Pressure Behavior

B9401 experiment did not measure the break flow, and the pressure behavior was only clue to judge whether the break flow was correctly calculated or not. Generally, break flow quality could vary according to the upstream conditions and depressurization characteristic through the break piping. Initially, the break flow was liquid single phase and the inventory loss was larger than other phase. As primary heat transport system pressure reduced and the vaporization was occurred, the break flow had vapor. As the void fraction of break flow increases, the break mass flowrate decreases due to decreasing mass flux.

The experiment started at 10 seconds as the p14 valve opened and RCP (Reactor Coolant Pump) and reactor trip occurred at 12 seconds. The break location was located in inlet header 8. After the break initiated at that time, the primary system pressure rapidly decreased as the inventory lost. Due to void generation, the slope of the depressurization rate decreased and few seconds later depressurization rate recovered as the ECI injection delivered into the HTS.

In view of depressurization rate, header depressurization is largely determined by break discharge rate and, later it is affected by ECC injection. Emergency coolant injection begins when the selected header pressure drops below the pre-determined injection pressure (header 7, 5.5 MPa). Header pressures determine when and at what flowrate the ECC flow enters each header. Since header 8 is the broken header, it has the fastest depressurization rate of the four headers during the blowdown. Header 6 is farthest from the broken header, and has the slowest depressurization rate. Header 7 is an outlet header, and has a depressurization rate between those of header 6 and header 8. Typically, in figure 3, the header pressure calculated by RELAP5, shows the reasonably good prediction in all headers and the above characteristics were reasonably well shown.

During the break, the primary pump speeds are reduced and ECC flow is initiated causing flows to change dramatically in the primary heat transport system. Header differential pressures (DP) provide an overall indication of flow directions in the below-header portion of the loop (inlet feeders, outlet feeders and heated sections) during the blowdown transient. In figure 7, RELAP5 predicted the differential pressure between headers reasonably. During this study, it was found that the minor pressure distribution trend differences in steady state would make relatively large differences in transient. In CANDU analysis, steady state pressure distribution should be treated more carefully than that of PWR (pressurized water reactor).

Primary Pumps Behavior

Primary loop coolant circulation was provided by two high-head centrifugal pumps. As mentioned earlier, the primary pumps were ramped down starting at 12 s. In figure 4,5,6, the oscillations were occurred, but the overall behavior was correctly predicted. In view of differential pressure (Figure 5), the primary pump 1 shows positive differential pressure during blowdown, and the flow direction was negative. On the other hand, the primary pump 2 shows negative differential pressure, and the flow direction was positive. These differential pressure histories of primary pump 1 and 2 behave correctly and good agreement of flow directions and amount of pressure differences across the pumps during the blowdown transient.

As shown in figure 4, pumps were coast down after pumps tripped. In figure 6, the pump1 outlet flow decreased due to the pump coast down, and the forces between driving force resulted from break mass flow and pump cost down force was balanced during a few seconds. The flow direction was reversed because the flow split was occurred near pump1 location. For this reason, the flow transient in this location was relatively mild compared to pump2. In the case of pump2, the flow was changed dramatically as the pump2 location was near the break (header 8). The pump2 outlet flow was maintained during the ECI injection was made because the injected ECI water was should spilt out through break but in experiment, the pump outlet flow showed very small flow. It seems that the differences arised from calculation and some errors should existed in calculation. On the other hand, in experiment, the flowmeter's measuring capability seemed to be limited by 30 L/sec. This limit is very small value to measure those large quantities. The calculation parameters except pump1 and pump2 outlet flows show good agreement with experiment. It seemed that the disagreements were resulted from the limitation of flow measurement device and calculation error.

Header ECI Flowrate behavior

In RD-14M and CANDU NPP, the ECI coolant delivered into each headers and the coolant could cool the heater section. For this reason, the timing and flowrate of ECC to each header are different. ECC flowrate to each header are important for analyzing ECC system behavior, and more importantly, for analyzing the fuel channel behavior.

Actually, ECI injection in RD-14M was actuated when header 7 pressure decrease below 5.5 MPa. After initiation of break at header 8, the header 7 pressure continuously decreased under 5.5MPa at 23.5 seconds. As shown in Figure 8, the calculated ECI flow behavior including injection signal generation time well predicted, but the initiation timing differences were shown in outlet header 5 and 7. In RD-14M calculation, the timings of ECI coolant delivery in each header were determined by the pressure distribution along the piping network including transient two-phase situation. These discrepancies might be arisen from the complicated effects, such as small two-phase pressure drop, header model itself, initial/transient pressure distribution in piping network, horizontal flow hydraulics and the predictability of steam condensation, etc.

In view of flow split, header 8 ECI flow was much larger than that of the others because the header was break header. The header 7 was much smaller than that of 5 and 6 as the ECI injection piping connected to header 7 and 8 were included in single branch. These flow-splitting behaviors among headers were well predicted.

FES (Fuel-Element-Simulator) Sheath Temperatures behavior of Heated Section

The maximum FES sheath temperature, usually called "Clad Temperature", is often the most important parameter in accident analyses. In test B9401, the maximum FES sheath temperatures occur in the high power channel of the critical (broken) pass, heated section 13 (HS13). The FES temperature excursions in HS13 began immediately at initiation of the break as flow in this channel dropped significantly to a very low value (stagnated channel). The FES temperatures initially rose quickly and then slowed as the heated channel power was reduced to decay levels beginning at about 12 s. Shortly after the onset of the high-pressure ECC injection phase, quenching began as ECC water arrived at the channel. The measured maximum FES temperature is that of the top pin in the middle of HS13. In figure 9, the calculated temperatures are not varies through the elevation but varied through the horizontal axis because RELAP5 cannot simulate heat structure elevation difference in the case that the horizontal heat structures exist in one hydraulic volume. Those behaviors can be observed in comparing "HS13-Top pin 6/6" and "HS13-Bottom pin 6/6" of figure 9. As described the above, in experiment, the maximum FES sheath temperature occurred in middle of fuel channel but the maximum temperature location was predicted as outlet. Maximum temperature also is different, 451°C (RELAP5) and 496.7 °C (B9401). It seems that these disagreements arised from the horizontal flow regime and heat transfer calculation, especially, horizontal quenching or reflooding phenomena.

Pressure Drop behavior across HS13

In the case of LOCA, the force balance between driving force from break and inertia including pressure loss makes a flow split phenomenon occur. In test B9401, a flow split occurs in at least some of the heated sections of the broken pass following the break. During the initial stage of the flow split, single-phase liquid flows out both ends of the heated section while rapid voiding of the channel occurs. Channel differential pressure, representatively HS13, provides an indication of the flow direction in the channel. In figure 10, HS13 differential pressure shows that calculated pressure has delayed about 10 seconds due to early initiation of ECI injection into header 5 but overall behavior agreed with experimental results well.

B. RELAP5/CANDU Results

In this section, RELAP5/CANDU computer code was used instead of RELAP5/MOD3.2 and the others including nodalization were exactly same as the RELAP5 case. In previous section, header pressure, primary pump, pressure drop across HS13, header ECI flowrate behavior, and FES (Fuel-Element-Simulator) sheath temperatures of heated section were discussed and the limited discussions were made in the area of significant differences.

In view of pressure and pump behavior, there were no differences between two cases because the models which affects to pressure behavior were not modified in RELAP5/CANDU code. In the case of fluid temperature, overall behavior showed good agreement with experiment but temperature transient became mild due to error correction in steam table. In Figure 11, the effect of the elevation of fuel rod was shown, but the differences were observed. This means that the heat transfer rate is larger than that of experiment and the heat transfer regime under stratified flow condition should be reviewed. In FES (Fuel Element Simulator) sheath temperature, the hydraulics inside channel was different. In figure 10, the RELAP5/CANDU calculation result shows that "horizontal stratified flow" regime appeared more frequently than that of RELAP5/MOD3.2. The "bubbly flow" regime was very frequently appeared during "horizontal stratified flow" due to the mass flux change. The transition criteria between "horizontal stratified flow" and "bubbly flow" regime were already considered in the course of channel model development but will be reviewed again more carefully.

In peak sheath temperature, 454.7°C was observed in exit nod of HS13. This temperature was slightly higher than that of RELAP5 but it was occurred in the same location. This means that the channel model did not affect to the temperature behavior in initial stage and the effect due to channel model may be shown in the later stage when the stratification was dominant. But the proper investigations were not made because the experiment had no significant temperature excursion in later stage.

C. Other Sensitivity Results

Feeder to End fitting connection Modeling

In view of pressure, the most significant difference in pressure loss under two-phase condition was nodalization in feeder connection. Originally, the junctions between feeders and test sections were modeled as normal junction without cross flow option but this nodalization predicted unreasonable pressure buildup at the middle of depressurization periods. In figure 4 shows that kind of behavior. In this study, in order to simulate the geometry as close as the real situation, the junctions were modeled by cross flow junction model that was neglected 'To-volume' momentum. In the case of normal junction, the pressure drop was adjusted by user input, 'Form Loss'. These two cases were not different during steady state calculation but, in transient under two-phase condition, the system behavior became different. Typically, figure 13 represent the differences among two calculations and experiment and the differences of depressurization started at around primary coolant saturation point. This means that the modeling differences let the depressurization characteristics change.

Break Upstream and Downstream Nodalization and break flow model

This nodalization changes were not so much differences because the system behaviors were governed by the driving forces arising from pressure gradient. Small difference of break flow was not much effective in overall system behavior.

V. CONCLUSIONS

RELAP5/MOD3.2 and RELAP5/CANDU simulations of the 30mm inlet header break test in the RD-14M multi channel facility have been performed, preliminarily with an aim to identify the RELAP5 applicability in a CANDU multi-channel system in comparison with the experimental results. The RELAP5/MOD3.2 predicted reasonably the main phenomena occurring in the transient. The conclusions from the present work are summarized as follows:

- 1) The RELAP5/MOD3 predicts reasonably overall thermal-hydraulic behaviors in the multi-channel inlet header break tests.
- 2) Pressure differences between headers govern the flow characteristics through the heated sections, particularly after the ECI. In determining header pressure, there are many uncertainties arisen from the complicated effects including steady state pressure distribution. Therefore, it would be concluded that further works are required to reduce these uncertainties, and consequently predict appropriately thermal-hydraulic behaviors in the reactor coolant system during LOCA analyses.
- RELAP5/MOD3 did not predict well the heater sheath temperature but in the case of RELAP5/CANDU, the predictability was improved because channel model was improved.
- 4) In B9401experiment, the stratification in channel and header was not enough dominant. Another experiment which the stratification occurred dominantly such as LOCA without ECI, will be analyzed and it is expected that RELAP5/CANDU will give more enhanced results.

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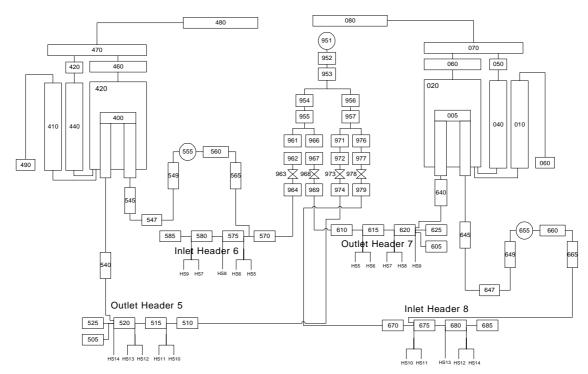


Figure 1. RD-14M Nodalization using RELAP5

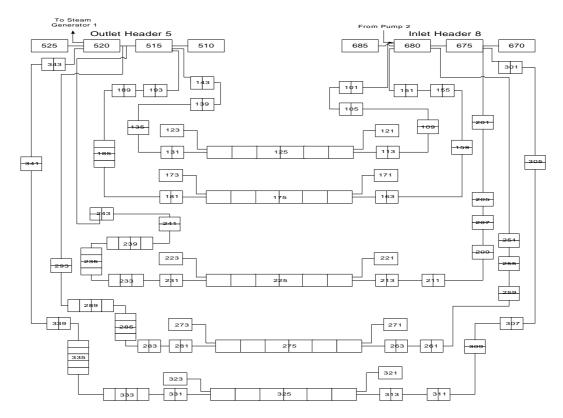


Figure 2. RD-14M Nodalization below Headers using RELAP5

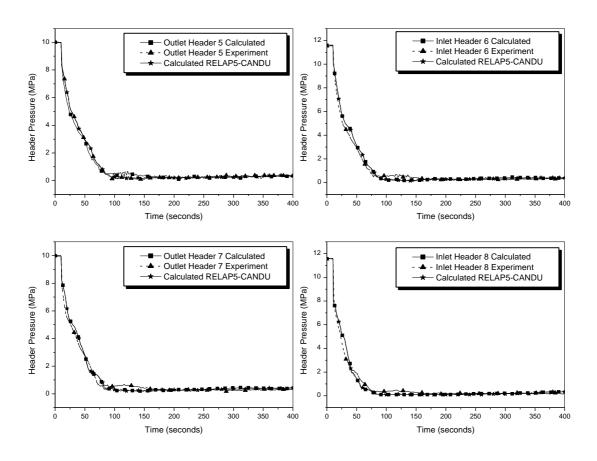
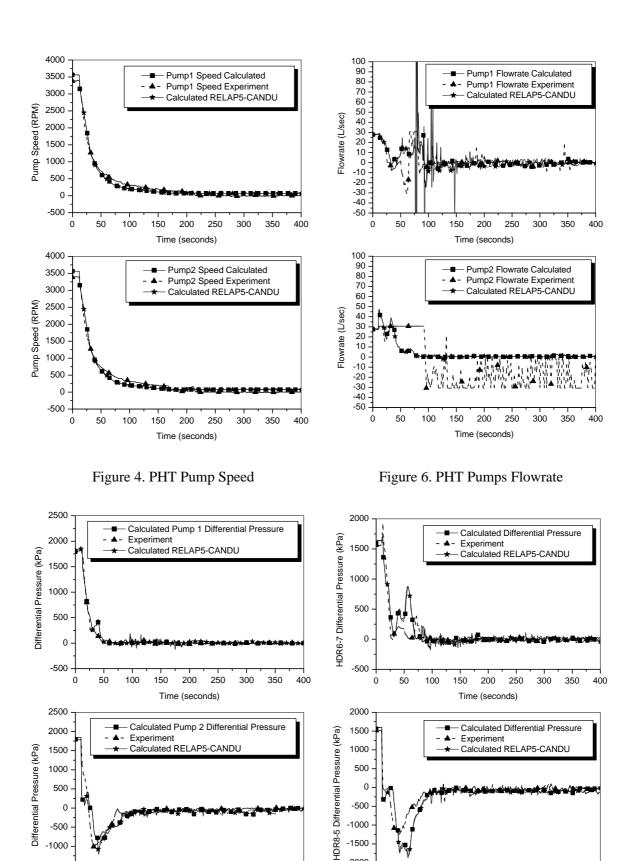


Figure 3. Header Pressure Behaviors



-500

-1000

-1500

-2000

Figure 5. PHT Pump Differential Pressure

Time (seconds)

-500

-1000

-1500

Figure 7. Header Differential Pressure

Time (seconds)

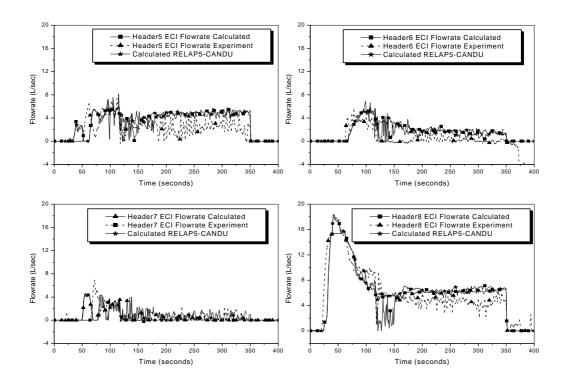


Figure 8. Headers ECI Flowrate

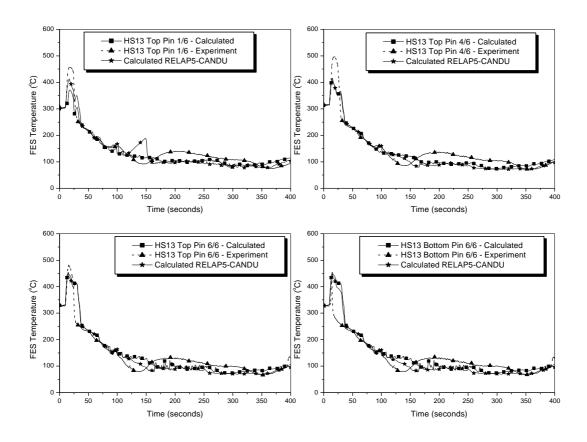
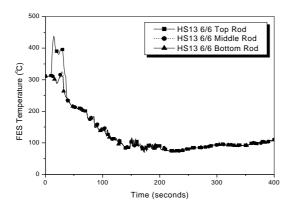


Figure 9. Channel 13 Sheath Temperature



HS13 Differential Pressure (kPa) -1.0x10 -1.5x10 50 100 150 200 250 350

B Differential Pressure Calculated
A Differential Pressure Experiment

1.5x10

5.0x10

-5.0x10

Figure 10. HS13 FES Temperature

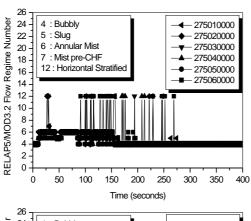
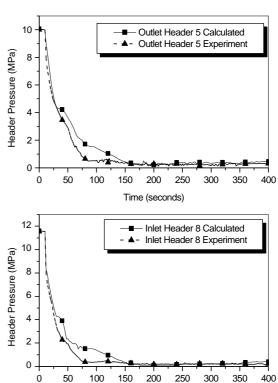


Figure 11. HS13 Differential Pressure



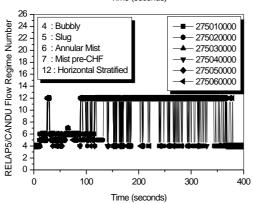


Figure 12. Flow Regime Comparison

Figure 13. Pressure without Cross Flow

Time (seconds)