

ANSYS 17×17

The Modal Analysis of the 17×17 Improved Fuel Assembly

493

SSE/LOCA

가 , 가 .
 - ,
 WECAN .
 ANSYS , ANSYS WECAN 17×17
 가
 WECAN ANSYS 가 , ANSYS
 WECAN 1% ,
 ANSYS .

Abstract

The fuel assembly SSE/LOCA analysis is performed to verify that the fuel assembly is limited to its stress criteria and maintained a coolable geometry and RCCA can be inserted in fuel assembly. The core model for this fuel assembly SSE/LOCA analysis is accomplished by simplified spring-mass model which is established using fuel assembly detailed model. Up to now, the fuel assembly detailed model has been established by using WECAN finite element code of Westinghouse Electric Co. In this study, to verify the ANSYS model in fuel assembly SSE/LOCA analysis, the 17×17 improved fuel assembly detailed models were established using the ANSYS and WECAN, and the modal analysis results for these models were compared. The difference of analysis results between ANSYS and WECAN model is less than 1%, therefore, ANSYS can be used in the fuel assembly SSE/LOCA analysis.

1.

SSE/LOCA Condition III IV 가
 ASME Section III NG 3000
 Appendix F (Rules for Evaluation of Service Loading with Level D Service Limits)

가 , 가
 , 가
 -
 WECAN
 [1] WECAN ANSYS
 [1] ANSYS
 17×17
 가 , ANSYS

2. Modal Analysis

2.1

가

$$\Pi = \int_{\Omega} \frac{1}{2} \varepsilon^T \sigma d\Omega - \int_{\Omega} u^T b d\Omega - \int_{\Gamma} u^T f d\Gamma \quad (1)$$

ε : (Strain) , σ : (Stress)
 u : (Displacement) , b : (Body Force)
 f : (Applied Force)

(Interpolation Function) (1)

$$\Pi_e = \int_{\Omega} \frac{1}{2} u^T (B^T DB) d\Omega - \int_{\Omega} u^T N^T b d\Omega - \int_{\Gamma} u^T N^T f d\Gamma \quad (2)$$

Π_e 가

$$\delta \Pi_e = \frac{\partial \Pi_e}{\partial u} = 0$$

(2)

$$K_e u = F_e \tag{3}$$

$$K_e = \int_{\Omega} (B^T D B) d\Omega, \quad F_e = \int_{\Omega} N^T b d\Omega - \int_{\Gamma} N^T f d\Gamma$$

(3)

2.2

가 Modal

$$M \ddot{u} + K u = 0 \tag{4}$$

Harmonic Equation (4)

Eigenvalue Equation

$$(K - \omega^2 M) u_0 = 0 \tag{5}$$

(5)가 Non-Trivial

$K - \omega^2 M$ Determinant가 0

$$|K - \lambda M| = 0, \quad \lambda = \omega^2 \tag{6}$$

Eigenvalue λ_i (Natural Frequency, $\omega_i = \sqrt{\lambda_i}$), Eigenvector u_i Mode Shape

3.

17×17, 1, 1, 가 6
 5, 가 1,
 1, 24, 1
 264
 ANSYS 17×17, ANSYS WECAN 17×17
 17×17

Effective Beam Mechanism, Beam Element, /
 2 Node-to-Node Contact Element
 /
 Sliding Element
 Effective, 17x17
 2 196
 372, ANSYS WECAN 가 ANSYS
 WECAN, Real Constant,

UX=0 @ node 2, 4, 175 & 176
 UY=0 @ node 176

Cold in Air Hot in Water Condition,

Master DOF, ANSYS Reduced Modal Analysis Option

4.

1 17x17
 가
 Real Constant 17x17 RFA 가
 17x17
 가, 1
 , WECAN ANSYS
 1% 3 4 17x17
 , WECAN ANSYS
 가
 , ANSYS 17x17
 WECAN 가

5.

ANSYS WECAN 17x17
 가
 , ANSYS WECAN
 1%

ANSYS

[1] , , , “ANSYS
”, 2004

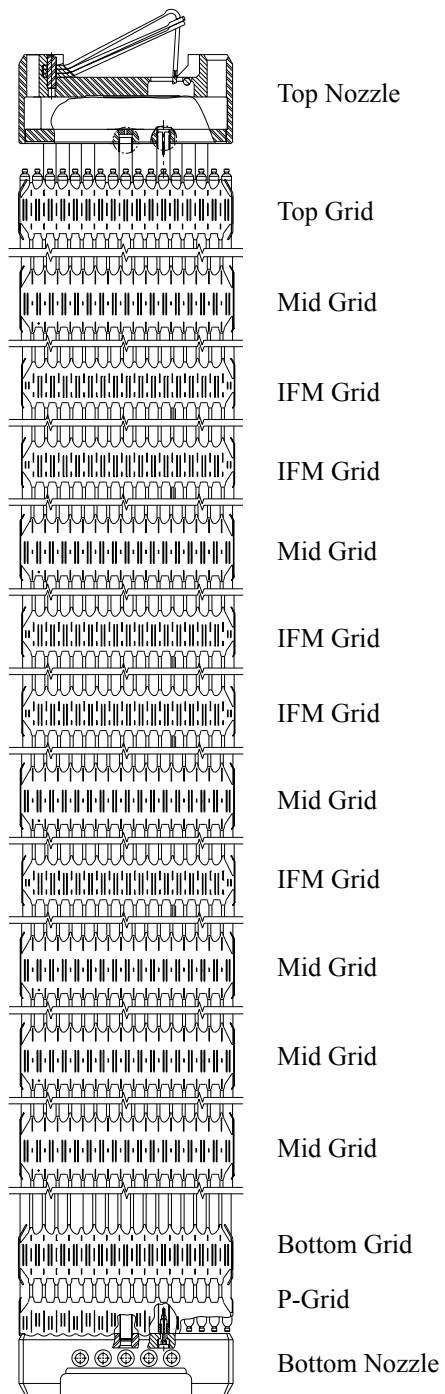
[2] WECAN User's Manual

[3] ANSYS User's Manual

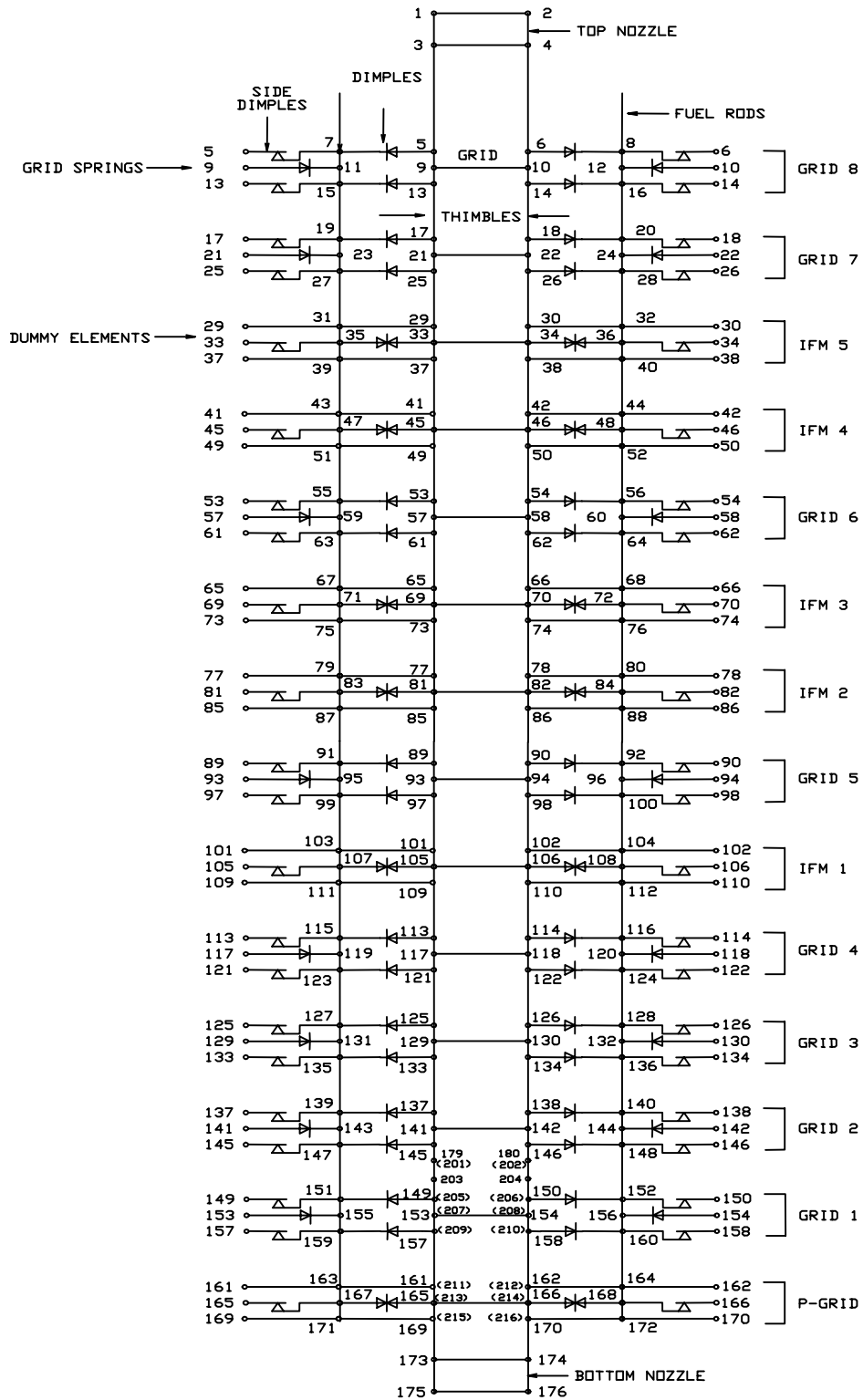
1. 17×17

Natural Frequency

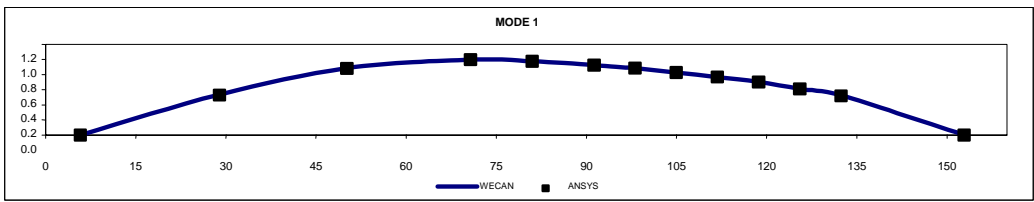
MODE	Cold in Air				Hot in Water		
	Test	WECAN	ANSYS	Ratio	WECAN	ANSYS	Ratio
1	3.69	3.82	3.82	1.000	3.29	3.29	1.000
2	8.08	8.18	8.18	1.000	7.04	7.05	0.999
3	12.93	13.24	13.25	0.999	11.40	11.40	1.000
4	18.81	19.63	19.64	0.999	16.87	16.88	0.999
5	24.61	26.11	26.12	1.000	22.42	22.42	1.000
6	32.30	38.17	38.16	1.000	32.76	32.75	1.000



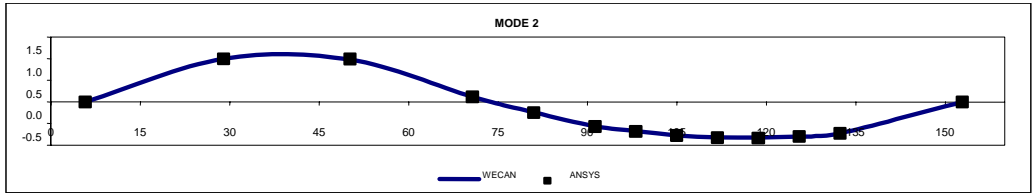
1. 17×17



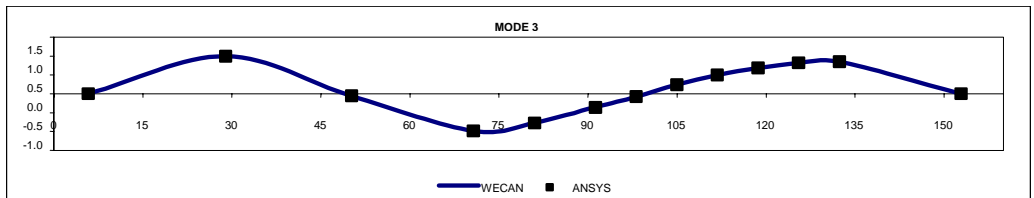
2. 17x17



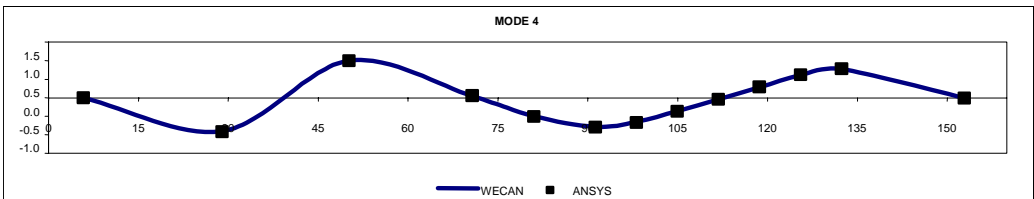
(a) 1st Mode



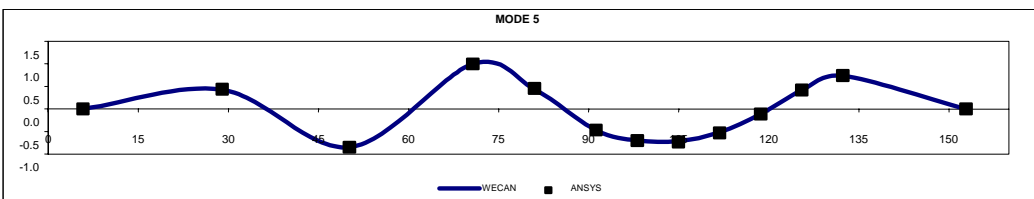
(b) 2nd Mode



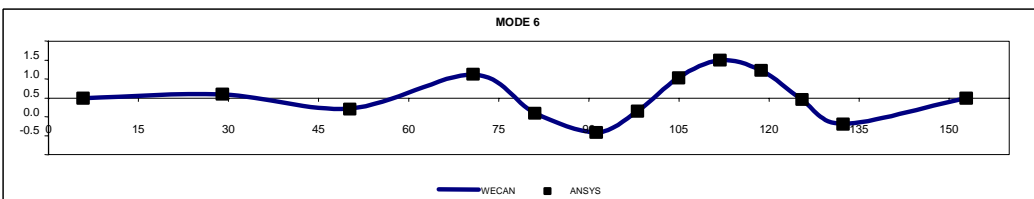
(c) 3rd Mode



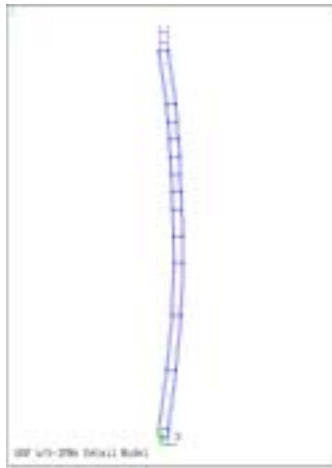
(d) 4th mode



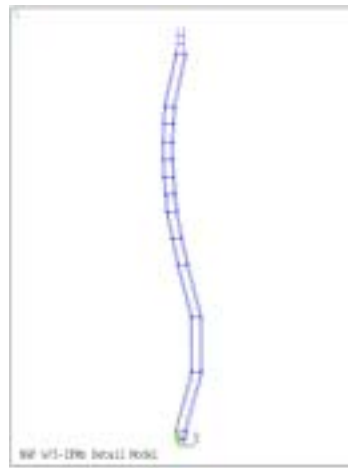
(e) 5th Mode



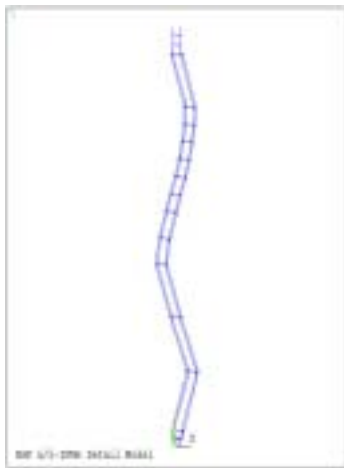
(f) 6th Mode



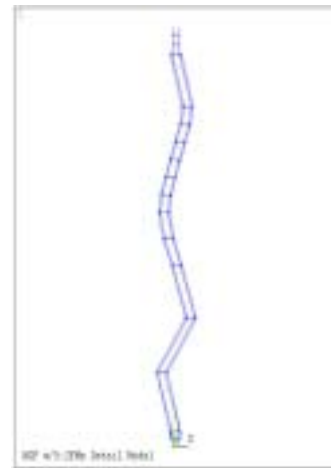
(a) 1st Mode



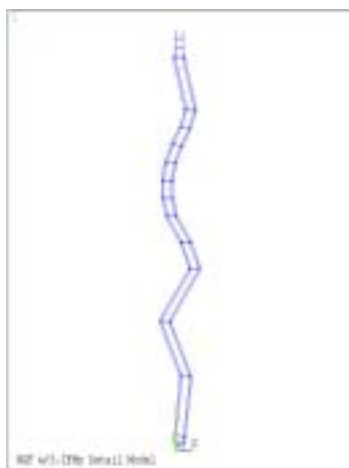
(b) 2nd Mode



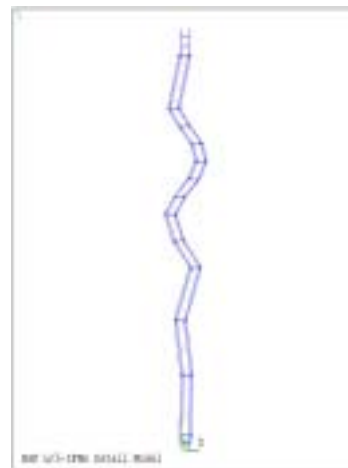
(c) 3rd Mode



(d) 4th Mode



(e) 5th Mode



(f) 6th Mode